

Permaglide® plain bearings

Catalogue 706



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Designs must only be prepared in accordance with the technical information, dimension tables and dimension drawings!

Due to constant development of the product range, we reserve the right to make modifications.

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706



Permaglide® plain bearings

With its new structure and pictograms, Catalogue 706 “Permaglide® Plain Bearings” will quickly guide you to the information you require.

It is subdivided into three main sections:

- Technical principles:
 - How is the rating life calculated?
 - How is a bearing arrangement designed?
 - How are Permaglide® plain bearings installed?
The section on “Technical principles” will provide the answers.
- Materials: This section contains information about the design, behaviour and areas of application of the Permaglide® materials.
- Designs and dimension tables:
 - This section contains the description and dimensions of the bearings offered in this catalogue.

The new structure is matched to the different stages involved in designing a bearing arrangement. Pictograms direct you to additional information.

This catalogue contains the standard Permaglide® range. The main dimensions conform to DIN ISO 3547 “Plain bearings. Wrapped bushes. Dimensions”.

Catalogue 706 has been completely revised and updated. It replaces Catalogue 705. Any information in earlier publications that does not agree with Catalogue 706 is therefore invalid.

If you would like to know more:

Using the CD-ROM “**medias**® professional” or the INA website at www.ina.com, you can for example calculate the rating life for Permaglide® plain bearings.

The publication ABP “Application Examples for Permaglide® Plain Bearings” shows exemplary design solutions.

Furthermore, our experts from the application engineering function and the INA engineering service are available to provide in-depth advice and prepare installation proposals.

INA-Schaeffler KG
Herzogenaurach

Permaglide® is a registered trademark and product of KS Gleitlager GmbH, St. Leon-Rot, Germany

Maintenance-free plain bearing material

primarily for dry running

Permaglide® P1

$$pV_{\max} = 1,8 \text{ N/mm}^2 \cdot \text{m/s}$$

$$pV_{\text{temp}} = 3,6 \text{ N/mm}^2 \cdot \text{m/s}$$

$$p_{\max \text{ stat.}} = 250 \text{ N/mm}^2$$

$$p_{\max \text{ dyn.}} = 56 \text{ N/mm}^2$$

$$v_{\max} = 2 \text{ m/s}$$

$$\vartheta = -200 \text{ }^\circ\text{C to } +280 \text{ }^\circ\text{C}$$

Low-maintenance plain bearing material

lubrication required

Permaglide® P2

$$pV_{\max} = 3 \text{ N/mm}^2 \cdot \text{m/s}$$

$$p_{\max \text{ stat.}} = 250 \text{ N/mm}^2$$

$$p_{\max \text{ dyn.}} = 70 \text{ N/mm}^2$$

$$v_{\max} = 3 \text{ m/s}$$

$$\vartheta = -40 \text{ }^\circ\text{C to } +110 \text{ }^\circ\text{C}$$

$$\vartheta_{\max} = \text{up to } +140 \text{ }^\circ\text{C for short periods}$$

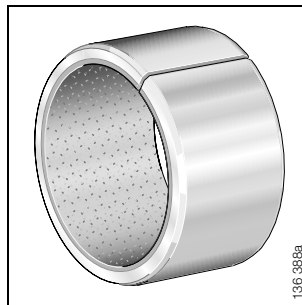
Bushes

maintenance-free:

PAP..P10

PAP..P11

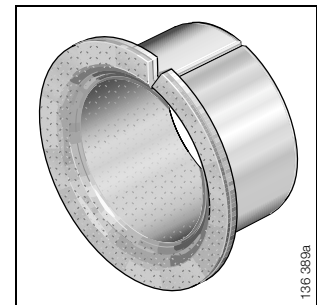
low-maintenance:
PAP..P20

**Flanged bushes**

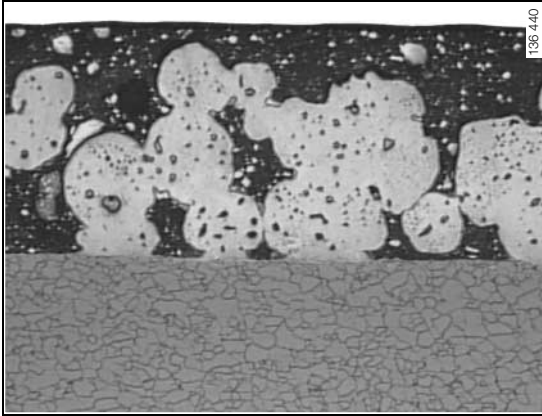
maintenance-free:

PAF..P10

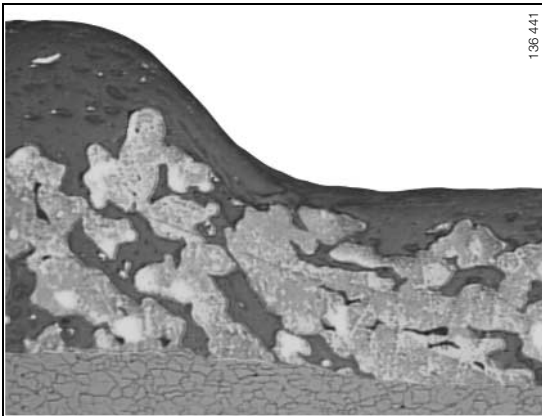
PAF..P11



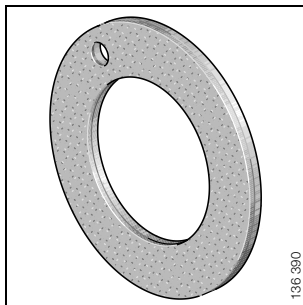
Permaglide® P1



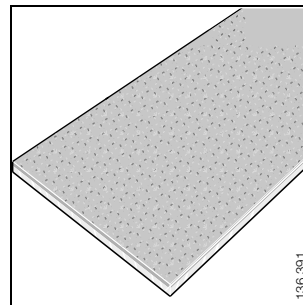
Permaglide® P2



Thrust washers PAW
maintenance-free:
PAW..P10
low-maintenance:
PAW..P20



Strips PAS
maintenance-free:
PAS..P10
PAS..P11
low-maintenance:
PAS..P20



Designs

Dimension tables

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US conversion factors

Surface roughness

ISO tolerances

Conversion factors

Dimensions	1 mm	0,039 in
	1 in	25,4 mm
	0,001 mm	0,00004 in
	0,001 in	0,025 mm
Mass	1 g	0,0022 lbs
	1 lb	453,6 g
Force	1 N	0,225 lbf
	1 lbf	4,45 N
Temperature	$^{\circ}\text{F} = \frac{9 \times ^{\circ}\text{C}}{5} + 32$	$^{\circ}\text{C} = \frac{5}{9} (^{\circ}\text{F} - 32)$
	-200 °C	-328 °F
	-40 °C	-40 °F
	+110 °C	+230 °F
	+140 °C	+284 °F
	+280 °C	+536 °F
Speed	1 m/s	196,848 fpm = 3,281 ft/s
	2 m/s	394 fpm
	3 m/s	590 fpm
	1 ft/s	0,3048 m/s
Torque	1 Nmm	0,009 in · lbf
	1 in · lbf	113 Nmm
	1 Nm	8,85 in · lbf
	1 in · lbf	0,113 Nm
Pressure	1 N/mm ² = 1 MPa	145 psi
	250 N/mm ²	36 258 psi
	1 psi	0,007 N/mm ² = 0,007 MPa
pv value	1,8 N/mm ² · m/s	51 390 psi · fpm
	3 N/mm ² · m/s	85 650 psi · fpm
	3,6 N/mm ² · m/s	102 780 psi · fpm

Surface roughness

R _a	AA and CLA	R _t	R _z	RMS	Surface symbols
μm	μinch	μm	μm	μinch	
0,2	8	1	1	8,96	▽▽▽▽
0,25	10	–	–	11,2	▽▽▽
0,3	12	1,5	1,6	13,44	
0,32	13	–	–	14,56	
0,4	16	2	2	17,92	
0,5	20	2,5	2,5	22,4	
0,63	25	3	3	28	▽▽
0,8	32	4	4	35,84	
1	40	–	–	44,8	
1,2	48	6,3	6,3	53,76	

ISO tolerances for shafts

Designation	Nominal dev.	Nominal dimension range in mm									
		over 3 incl. 6	6 10	10 18	18 30	30 50	50 80	80 120	120 180	180 250	250 315
		Deviations in μm									
f 7	upper	-10	-13	-16	-20	-25	-30	-36	-43	-50	-56
	lower	-22	-28	-34	-41	-50	-60	-71	-83	-96	-108
h 6	upper	0	0	0	0	0	0	0	0	0	0
	lower	-8	-9	-11	-13	-16	-19	-22	-25	-29	-32
h 7	upper	0	0	0	0	0	0	0	0	0	0
	lower	-12	-15	-18	-21	-25	-30	-35	-40	-46	-52
h 8	upper	0	0	0	0	0	0	0	0	0	0
	lower	-18	-22	-27	-33	-39	-46	-54	-63	-72	-81

ISO tolerances for holes

Designation	Nominal dev.	Nominal dimension range in mm									
		over 3 incl. 6	6 10	10 18	18 30	30 50	50 80	80 120	120 180	180 250	250 315
		Deviations in μm									
G 7	upper	+16	+20	+24	+28	+34	+40	+47	+54	+61	+69
	lower	+4	+5	+6	+7	+9	+10	+12	+14	+15	+17
H 6	upper	+8	+9	+11	+13	+16	+19	+22	+25	+29	+32
	lower	0	0	0	0	0	0	0	0	0	0
H 7	upper	+12	+15	+18	+21	+25	+30	+35	+40	+46	+52
	lower	0	0	0	0	0	0	0	0	0	0
H 8	upper	+18	+22	+27	+33	+39	+46	+54	+63	+72	+81
	lower	0	0	0	0	0	0	0	0	0	0
J 7	upper	+6	+8	+10	+12	+14	+18	+22	+26	+30	+36
	lower	-6	-7	-8	-9	-11	-12	-13	-14	-16	-16

Symbols and units

Unless stated otherwise in the text, the values used in this catalogue have the following symbols, units and definitions.

A	mm ²	Outside surface area of bush
B	mm	Width of bush, total width of strip
B ₁	mm	Usable width of strip
C _i	mm	Inner edge break
C _o	mm	Outer chamfer
D _{FL}	mm	Flange diameter
D _i	mm	Inside diameter of bush Inside diameter of thrust washer
D _{iE}	mm	Inside diameter of bush after pressing in
D _o	mm	Outside diameter of bush Outside diameter of thrust washer
d ₁	mm	Diameter of fixing hole in thrust washer
d _{6a}	mm	Diameter of housing recess for thrust washers
d _G	mm	Diameter of housing bore
d _H	mm	Inside diameter of auxiliary ring
d _K	mm	Diameter of burnishing tool
d _L	mm	Diameter of lubrication hole in PAP..P20 bush
d _W	mm	Shaft diameter
E _G	N/mm ²	Modulus of elasticity of housing
E _L	N/mm ²	Modulus of elasticity of bearing
F	N	Bearing load, press-in force
f _G	mm	Chamfer width on housing
f _A	–	Correction factor for loading
f _L	–	Correction factor for linear movement
f _p	–	Correction factor for load
f _R	–	Correction factor for roughness depth
f _v	–	Correction factor for sliding speed
f _W	–	Correction factor for material
f _θ	–	Correction factor for temperature
H	mm	Stroke length for linear movement

Symbols and units

J	mm	Pitch circle diameter of thrust washer
L	mm	Length of strip
L_h	h	Basic rating life
m	g	Mass
n	min^{-1}	Speed
n_{osc}	min^{-1}	Oscillation frequency of to and fro movement
p	N/mm^2	Specific bearing load
p_1	N/mm^2	Joint pressure
p_v	$\text{N}/\text{mm}^2 \cdot \text{m}/\text{s}$	p_v value: product of the specific bearing load and the sliding speed
R	mm	Radius
R_{rel}	$\Omega \cdot \text{cm}^2$	Relative electrical resistance
R_z	μm	Roughness depth
R_{zG}	μm	Roughness depth of housing bore
R_{zL}	μm	Roughness depth of bush backing material
s_1	mm	Thickness of steel or bronze backing
s_3	mm	Wall thickness of bush
s_{FL}	mm	Thickness of flange
s_G	mm	Wall thickness of housing
s_{Mat}	mm	Material removal during running-in
s_4	mm	Machining allowance
t_a	mm	Depth of the housing recess
v	m/s	Sliding speed
U	mm	Interference

α_{Bz}	K^{-1}	Coefficient of thermal expansion for bronze
α_{St}	K^{-1}	Coefficient of thermal expansion for steel
Δs	mm	Theoretical internal clearance
ϑ	$^{\circ}C, K$	Operating temperature
λ_{Bz}	$W/(m \cdot K)$	Coefficient of thermal conductivity for bronze
λ_{St}	$W/(m \cdot K)$	Coefficient of thermal conductivity for steel
μ	–	Coefficient of friction
μ_L	–	Coefficient of friction between bush backing and housing bore
ν_G	–	Transversal contraction of housing
ν_L	–	Transversal contraction of bearing
φ	$^{\circ}$	Oscillation angle

Rating life

Operating life

The operating life is the life actually achieved by a plain bearing. This can deviate substantially from the calculated basic rating life.

Calculation of rating life

Calculation of the basic rating life applies to plain bearings with

- rotary movement
- oscillating movement
- linear movement
 - please consult the INA engineering service for Permaglide® P2.

The basic rating life is essentially dependent on

- the pv value
 - specific bearing load
 - sliding speed
- mating surface
 - material
 - roughness depth
 - surface structure
- operating temperature.

It is not possible to quantify the following in mathematical terms:

- corrosion – in dry running of Permaglide® P1
- lubricant ageing – in grease lubrication of Permaglide® P2
- contamination.

Calculated rating life – guide value

For the reasons stated above, the calculated basic rating life can serve only as a guide value.

Unrealistic guide values will result under the following conditions:

- very low bearing loads or
- very low sliding speeds.



The range (Table 1) only indicates the limits within which it is realistic to carry out life calculation.

The Permaglide® plain bearing material can be loaded up to the values given in Table 2.

The rating life may be shortened or extended by special operating conditions: guide values are given in Table 3.



Range

Table 1 · Range of the life calculation

Range for	Permaglide® P1	Permaglide® P2
pv value		
pv N/mm ² · m/s	0,03 ≤ pv ≤ 1,8	0,2 ≤ pv ≤ 3
Specific load		
p N/mm ²	p ≤ 56	p ≤ 70
Sliding speed		
v m/s	v ≤ 2	v ≤ 3

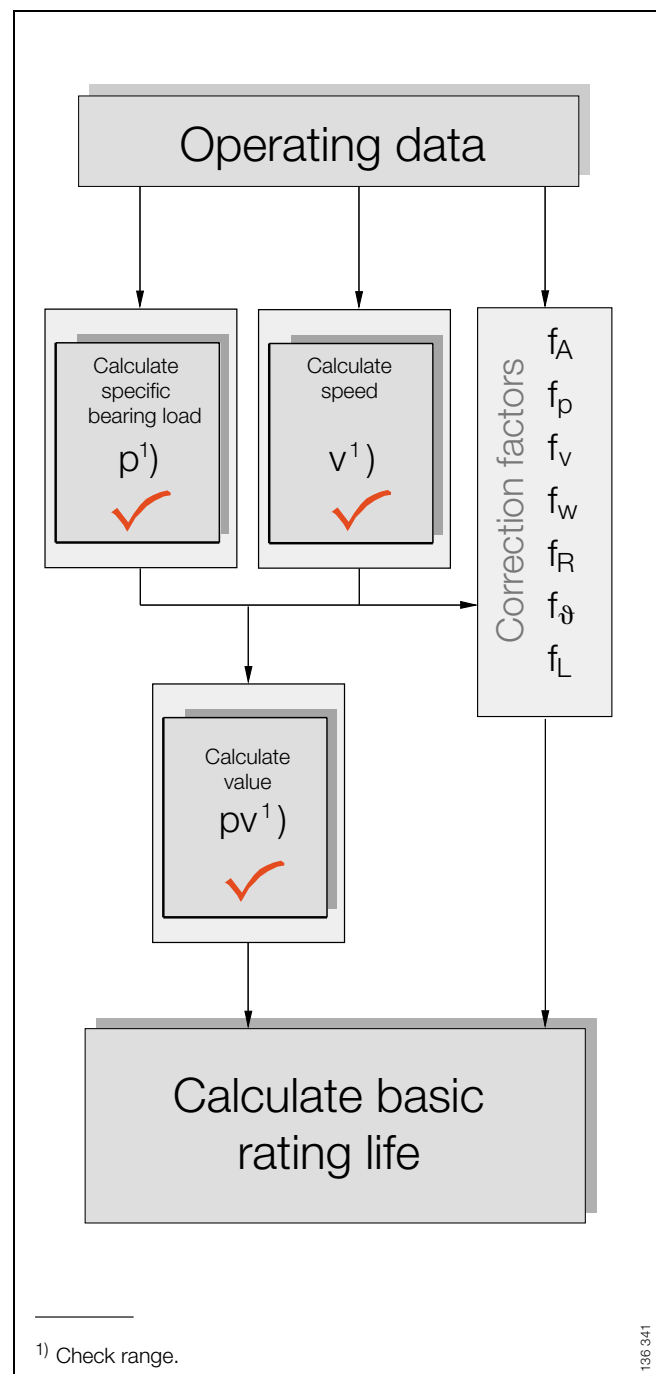
Table 2 · Permissible loads

Permissible loads	Permaglide® P1	Permaglide® P2
pv value		
pv _{max} N/mm ² · m/s	1,8	3
pv ¹⁾ N/mm ² · m/s	3,6	-
Specific load p		
p _{max stat.} N/mm ²	250	250
p _{max} ²⁾ N/mm ²	140	140
p _{max dyn.} N/mm ²	56	70
Sliding speed v		
v _{max} m/s	2	3
Operating temperature ϑ		
ϑ °C	-200 to +280	-40 to +110
$\vartheta_{max}^{1)}$ °C	-	to +140

- 1) For short periods.
- 2) Very low sliding speed.

Table 3 · Guide values for the rating life of P1 under special operating conditions

Operating conditions	Rating life
Dry running, intermittent	200% L _h
Alternating between dry running and running in water	20% L _h
Running in water	200% L _h
Continuous operation in fluid lubricants	300% L _h
Continuous operation in lubricating greases	50% to 150% L _h



1) Check range.

136 341

Figure 1 · Life calculation diagram

Rating life

Basic rating life

Maintenance-free Permaglide® P1

Rotary movement

$$L_h = \frac{400}{(pv)^{1,2}} \cdot f_A \cdot f_p \cdot f_v \cdot f_{\vartheta} \cdot f_W \cdot f_R \quad (1)$$

Linear movement

$$L_h = \frac{400}{(pv)^{1,2}} \cdot f_A \cdot f_p \cdot f_v \cdot f_{\vartheta} \cdot f_W \cdot f_R \cdot f_L \quad (2)$$

Low-maintenance Permaglide® P2

Rotary movement

$$L_h = \frac{2000}{(pv)^{1,5}} \cdot f_A \cdot f_p \cdot f_v \cdot f_{\vartheta} \cdot f_W \cdot f_R \quad (3)$$

Specific bearing load

Bush

$$p = \frac{F}{D_i \cdot B} \quad (4)$$

Thrust washer

$$p = \frac{4 \cdot F}{(D_o^2 - D_i^2) \cdot \pi} \quad (5)$$

Sliding speed

Bush, rotary movement

$$v = \frac{D_i \cdot \pi \cdot n}{60 \cdot 10^3} \quad (6)$$

Bush, oscillating movement


$$v = \frac{D_i \cdot \pi}{60 \cdot 10^3} \cdot \frac{2\varphi \cdot n_{osc}}{360^\circ} \quad (7)$$

Thrust washer, rotary movement

$$v = \frac{D_o \cdot \pi \cdot n}{60 \cdot 10^3} \quad (8)$$

Thrust washer, oscillating movement

$$v = \frac{D_o \cdot \pi}{60 \cdot 10^3} \cdot \frac{2\varphi \cdot n_{osc}}{360^\circ} \quad (9)$$

 Check whether p, v and pv are within range – see Table 1, page 11.



- B mm
Width of bush, see *dimension tables*
- D_i mm
Inside diameter of bush, see *dimension tables*
Inside diameter of thrust washer, see *dimension tables*
- D_o mm
Outside diameter of thrust washer, see *dimension tables*
- F N
Bearing load
- f_A –
Correction factor for loading, Figure 3, page 14
- f_p –
Correction factor for load, Figure 4, page 15
- f_v –
Correction factor for sliding speed, Figure 5, page 15
- f_θ –
Correction factor for temperature, Figure 6, page 15
- f_W –
Correction factor for material, Table 4, page 14
- f_R –
Correction factor for roughness depth, Figure 7, page 15
- f_L –
Correction factor for linear movement, see page 16
- L_h h
Basic rating life
- n min^{-1}
Speed
- n_{osc} min^{-1}
Oscillation frequency, Figure 2
- p N/mm^2
Specific bearing load
- v m/s
Sliding speed
- φ °
Oscillation angle, Figure 2.

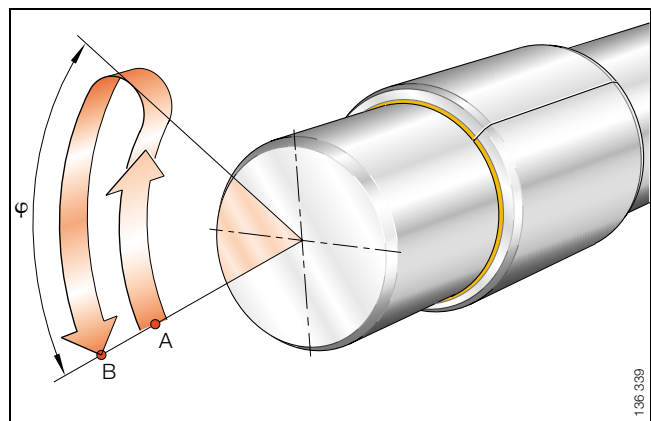


Figure 2 · Oscillation angle φ
The oscillation frequency n_{osc} is the number of movements from A to B per minute

Correction factors

- Correction factor for loading f_A , Figure 3
 - Point load $f_A = 1$
Rotating shaft, stationary bush
 - Circumferential load $f_A = 2$
Stationary shaft, rotating bush
 - Thrust washer $f_A = 1$
 - Linear movement $f_A = 1$
- Correction factor for load f_p , Figure 4, page 15
- Correction factor for sliding speed f_v , Figure 5, page 15
- Correction factor for temperature f_θ , Figure 6, page 15
- Correction factor for material f_W , Table 4, page 14
- Correction factor for roughness depth f_R , Figure 7, page 15
- Correction factor for linear movement f_L , page 16.

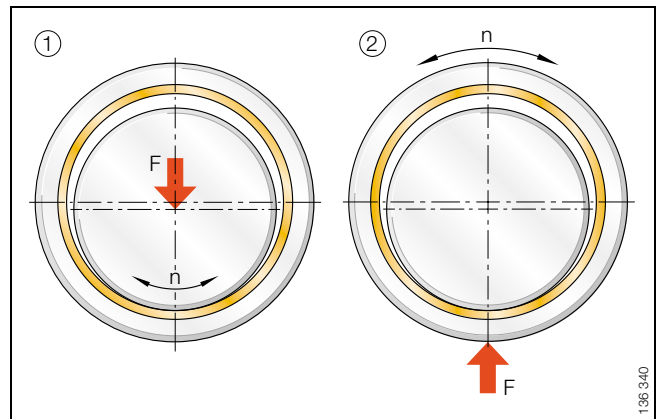


Figure 3 · Correction factor for loading f_A
 ① Point load $f_A = 1$
 ② Circumferential load $f_A = 2$

Table 4 · Correction factor for material f_W with roughness depth R_z2 to R_z3 of the mating surface for Permaglide® P1 and P2

Mating surface material	f_W
Steel	1
Nitrided steel	1
Corrosion-resistant steel	2
Hard-chromium plated steel (layer thickness min. 0,013 mm)	2
Zinc-plated steel (layer thickness min. 0,013 mm)	0,2
Phosphated steel (layer thickness min. 0,013 mm)	0,2
Cast iron R_z2	1
Anodized aluminium	0,4
Hard anodized aluminium (hardness 450 +50 HV; 0,025 mm thick)	2
Copper-based alloys	0,1 to 0,4
Nickel	0,2

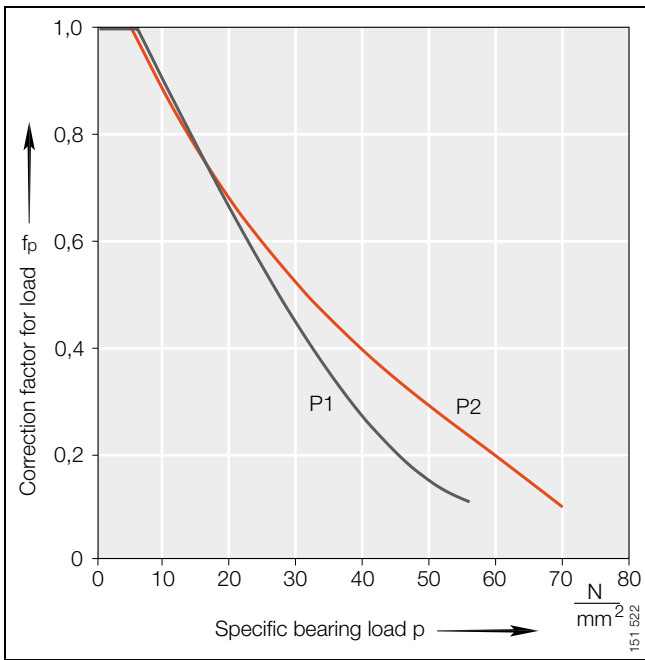


Figure 4 · Correction factor for load f_p ,
Permaglide® P1, P2

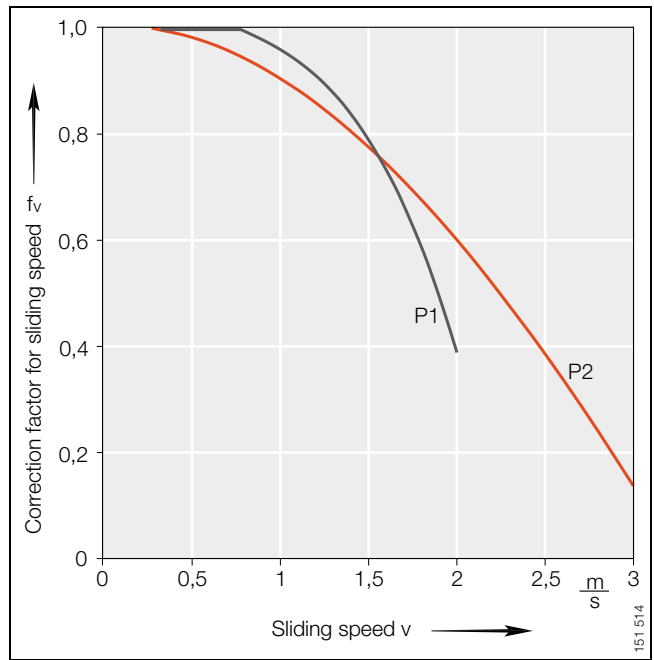


Figure 5 · Correction factor for sliding speed f_v ,
Permaglide® P1, P2

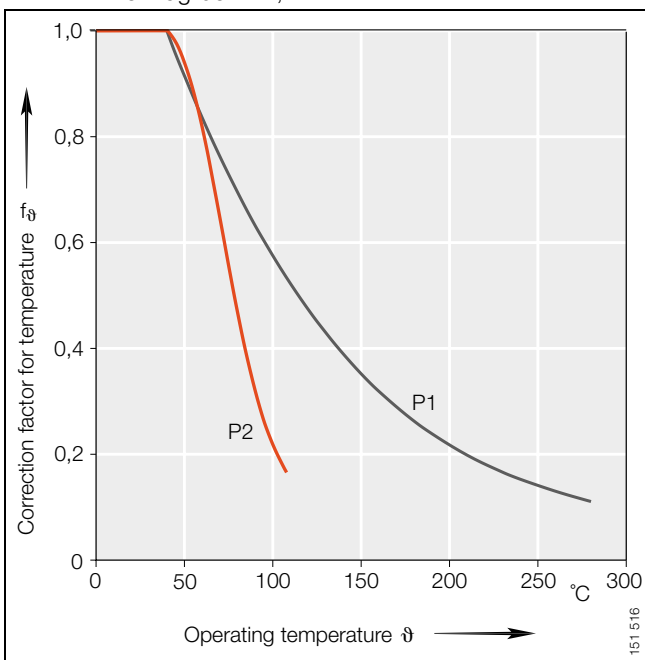


Figure 6 · Correction factor for temperature f_θ ,
Permaglide® P1, P2

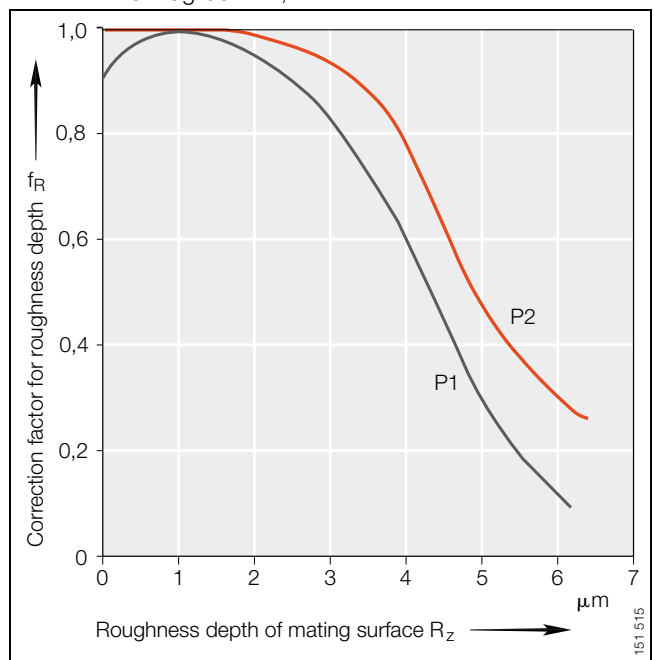


Figure 7 · Correction factor for roughness depth f_R ,
Permaglide® P1, P2

Rating life

Calculation of correction factor f_L for Permaglide® P1:

$$f_L = 0,65 \cdot \frac{B}{H+B} \quad (10)$$

f_L -
Correction factor for linear movement

B mm
Width of bush, see *dimension tables*

H mm
Stroke length.

Stroke length for linear movement, Figure 8:

⚠ Limit value: $H_{\max} = 2,5 \times B$.

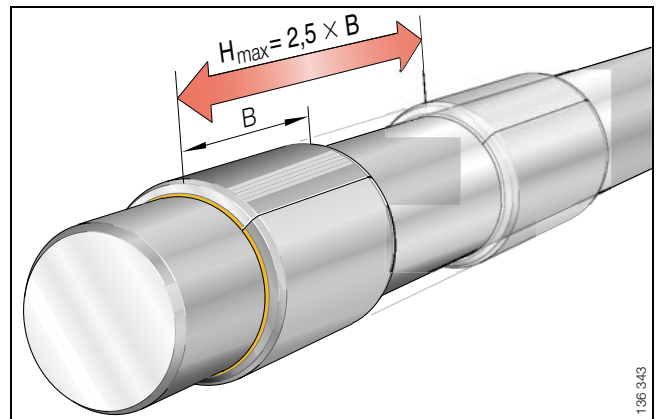


Figure 8 · Linear movement, stroke length H_{\max}

Rating life

Calculation example
Bush PAP..P10



A bearing with a rating life of 1000 h under point load is required for a steel shaft of 20 mm diameter

Bush PAP 2015 P10 Operating data	Inside diameter of bush $D_i = 20 \text{ mm}$	Width of bush $B = 15 \text{ mm}$	Bearing load $F = 300 \text{ N}$	Speed $n = 500 \text{ min}^{-1}$	Correction factors
Required					
Specific bearing load p <i>Check range!</i>	$p = \frac{F}{D_i \cdot B} = \frac{300}{20 \cdot 15} \text{ N/mm}^2$		$p = 1 \text{ N/mm}^2$		
Sliding speed v <i>Check range!</i>	$v = \frac{D_i \cdot \pi \cdot n}{60 \cdot 10^3} = \frac{20 \cdot \pi \cdot 500}{60 \cdot 10^3} \text{ m/s}$		$v = 0,52 \text{ m/s}$		
pv value <i>Check range!</i>	$pv = p \cdot v = 1 \cdot 0,52 \text{ N/mm}^2 \cdot \text{m/s}$		$pv = 0,52 \text{ N/mm}^2 \cdot \text{m/s}$		
Correction factor for loading f_A	$f_A = 1$	Point load	Figure 3, page 14		
Correction factor for load f_p	$f_p = 1$		Figure 4, page 15		
Correction factor for sliding speed f_v	$f_v = 1$		Figure 5, page 15		
Correction factor for temperature f_ϑ	$f_\vartheta = 1$	$\vartheta = +35 \text{ }^\circ\text{C}$	Figure 6, page 15		
Correction factor for material f_W	$f_W = 1$	Steel	Table 4, page 14		
Correction factor for roughness depth f_R	$f_R = 0,96$	$R_z = 2$	Figure 7, page 15		
Basic rating life L_h	$L_h = \frac{100}{(pv)^{1,2}} \cdot f_A \cdot f_p \cdot f_v \cdot f_\vartheta \cdot f_W \cdot f_R = \frac{400}{0,52^{1,2}} \cdot 1 \cdot 1 \cdot 1 \cdot 1 \cdot 1 \cdot 0,96 \text{ h}$				
	$L_h = 842 \text{ h} < 1000 \text{ h} = L_{h \text{ required}}$		Condition not fulfilled		

New calculation with wider bush PAP 2020 P10

Bush PAP 2020 P10 Operating data	Inside diameter of bush $D_i = 20 \text{ mm}$	Width of bush $B = 20 \text{ mm}$	Bearing load $F = 300 \text{ N}$	Speed $n = 500 \text{ min}^{-1}$	Correction factors
Required					
Specific bearing load p <i>Check range!</i>	$p = \frac{F}{D_i \cdot B} = \frac{300}{20 \cdot 20} \text{ N/mm}^2$		$p = 0,75 \text{ N/mm}^2$		
Sliding speed v <i>Check range!</i>	$v = \frac{D_i \cdot \pi \cdot n}{60 \cdot 10^3} = \frac{20 \cdot \pi \cdot 500}{60 \cdot 10^3} \text{ m/s}$		$v = 0,52 \text{ m/s}$		
pv value <i>Check range!</i>	$pv = p \cdot v = 0,75 \cdot 0,52 \text{ N/mm}^2 \cdot \text{m/s}$		$pv = 0,39 \text{ N/mm}^2 \cdot \text{m/s}$		
Correction factor for loading f_A	$f_A = 1$	Point load	Figure 3, page 14		
Correction factor for load f_p	$f_p = 1$		Figure 4, page 15		
Correction factor for sliding speed f_v	$f_v = 1$		Figure 5, page 15		
Correction factor for temperature f_ϑ	$f_\vartheta = 1$	$\vartheta = +35 \text{ }^\circ\text{C}$	Figure 6, page 15		
Correction factor for material f_W	$f_W = 1$	Steel	Table 4, page 14		
Correction factor for roughness depth f_R	$f_R = 0,96$	$R_z = 2$	Figure 7, page 15		
Basic rating life L_h	$L_h = \frac{100}{(pv)^{1,2}} \cdot f_A \cdot f_p \cdot f_v \cdot f_\vartheta \cdot f_W \cdot f_R = \frac{400}{0,39^{1,2}} \cdot 1 \cdot 1 \cdot 1 \cdot 1 \cdot 1 \cdot 0,96 \text{ h}$				
	$L_h = 1189 \text{ h} > 1000 \text{ h} = L_{h \text{ required}}$		Selected: Bush PAP 2020 P10		

Rating life

Calculation example
Thrust washer PAW..P20

A rating life of 4 000 h is required for a thrust washer under an axial load of 1500 N

Thrust washer PAW 28 P20 Operating data	Inside diameter of thrust washer $D_i = 28 \text{ mm}$	Outside diameter of thrust washer $D_o = 48 \text{ mm}$	Bearing load $F = 1500 \text{ N}$	Speed $n = 200 \text{ min}^{-1}$	Correction factors
Required					
Specific bearing load p <i>Check range!</i>	$p = \frac{4 \cdot F}{(D_o^2 - D_i^2) \cdot \pi} = \frac{4 \cdot 1500}{(48^2 - 28^2) \cdot \pi}$		$p = 1,26 \text{ N/mm}^2$		
Sliding speed v <i>Check range!</i>	$v = \frac{D_o \cdot \pi \cdot n}{60 \cdot 10^3} = \frac{48 \cdot \pi \cdot 200}{60 \cdot 10^3} \text{ m/s}$		$v = 0,5 \text{ m/s}$		
pv value <i>Check range!</i>	$pv = p \cdot v = 1,26 \cdot 0,5 \text{ N/mm}^2 \cdot \text{m/s}$		$pv = 0,63 \text{ N/mm}^2 \cdot \text{m/s}$		
Correction factor for loading f_A	$f_A = 1$	Axial load	Figure 3, page 14		
Correction factor for load f_p	$f_p = 1$		Figure 4, page 15		
Correction factor for sliding speed f_v	$f_v = 0,98$		Figure 5, page 15		
Correction factor for temperature f_ϑ	$f_\vartheta = 1$	$\vartheta = +20 \text{ }^\circ\text{C}$	Figure 6, page 15		
Correction factor for material f_W	$f_W = 1$	Steel	Table 4, page 14		
Correction factor for roughness depth f_R	$f_R = 0,98$	$R_z = 2$	Figure 7, page 15		
Basic rating life L_h	$L_h = \frac{2000}{(pv)^{1,5}} \cdot f_A \cdot f_p \cdot f_v \cdot f_\vartheta \cdot f_W \cdot f_R = \frac{2000}{0,63^{1,5}} \cdot 1 \cdot 1 \cdot 0,98 \cdot 1 \cdot 1 \cdot 0,98 \text{ h}$				
$L_h = 3841 \text{ h} < 4000 \text{ h} = L_{h \text{ required}}$ Condition not fulfilled					

New calculation with larger thrust washer PAW 32 P20

Thrust washer PAW 32 P20 Operating data	Inside diameter of thrust washer $D_i = 32 \text{ mm}$	Outside diameter of thrust washer $D_o = 54 \text{ mm}$	Bearing load $F = 1500 \text{ N}$	Speed $n = 200 \text{ min}^{-1}$	Correction factors
Required					
Specific bearing load p <i>Check range!</i>	$p = \frac{4 \cdot F}{(D_o^2 - D_i^2) \cdot \pi} = \frac{4 \cdot 1500}{(54^2 - 32^2) \cdot \pi}$		$p = 1,01 \text{ N/mm}^2$		
Sliding speed v <i>Check range!</i>	$v = \frac{D_o \cdot \pi \cdot n}{60 \cdot 10^3} = \frac{54 \cdot \pi \cdot 200}{60 \cdot 10^3} \text{ m/s}$		$v = 0,57 \text{ m/s}$		
pv value <i>Check range!</i>	$pv = p \cdot v = 1,01 \cdot 0,57 \text{ N/mm}^2 \cdot \text{m/s}$		$pv = 0,58 \text{ N/mm}^2 \cdot \text{m/s}$		
Correction factor for loading f_A	$f_A = 1$	Axial load	Figure 3, page 14		
Correction factor for load f_p	$f_p = 1$		Figure 4, page 15		
Correction factor for sliding speed f_v	$f_v = 0,97$		Figure 5, page 15		
Correction factor for temperature f_ϑ	$f_\vartheta = 1$	$\vartheta = +20 \text{ }^\circ\text{C}$	Figure 6, page 15		
Correction factor for material f_W	$f_W = 1$	Steel	Table 4, page 14		
Correction factor for roughness depth f_R	$f_R = 0,98$	$R_z = 2$	Figure 7, page 15		
Basic rating life L_h	$L_h = \frac{2000}{(pv)^{1,5}} \cdot f_A \cdot f_p \cdot f_v \cdot f_\vartheta \cdot f_W \cdot f_R = \frac{2000}{0,58^{1,5}} \cdot 1 \cdot 1 \cdot 0,97 \cdot 1 \cdot 1 \cdot 0,98 \text{ h}$				
$L_h = 4304 \text{ h} > 4000 \text{ h} = L_{h \text{ required}}$ Selected: Thrust washer PAW 32 P20					

Design of bearing arrangements



Housing

Bushes

Permaglide® bushes are pressed into the housing. This provides both axial and radial location. No further means of fixing are required.

The following are recommended for the housing bore:

- roughness depth R_z10
- chamfer $f_G \times 20^\circ \pm 5^\circ$ (Figure 9, Table 5). This chamfer makes it easier to press the bush into the housing.

Table 5 · Chamfer width f_G on the housing bore for bushes (Figure 9)

Bore diameter d_G	Chamfer width f_G
$d_G \leq 30$	$0,8 \pm 0,3$
$30 < d_G \leq 80$	$1,2 \pm 0,4$
$80 < d_G \leq 180$	$1,8 \pm 0,8$
$180 < d_G$	$2,5 \pm 1,0$

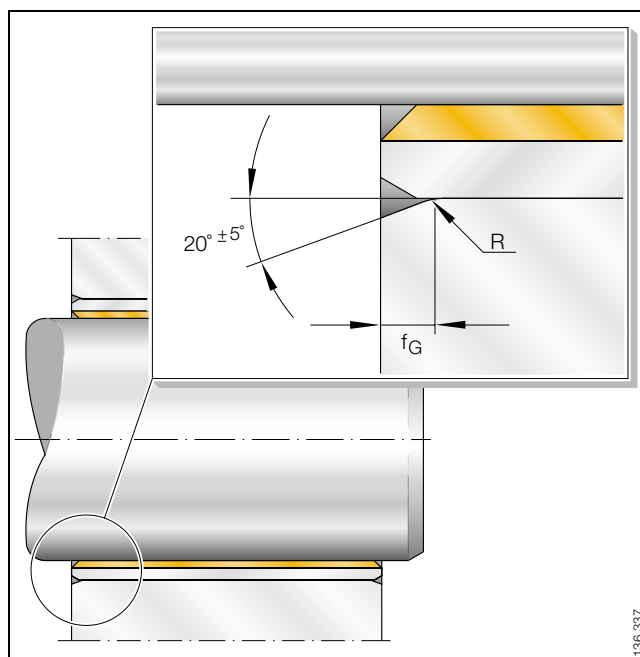


Figure 9 · Chamfer on housing for bush PAP

Flanged bushes

The radius at the transition from the radial to the axial component must be taken into consideration for flanged bushes (Figure 10, Table 6).

- the flanged bush must not be in contact in the area of the radius
- in applications with axial loads, the flange must be adequately supported.

Table 6 · Chamfer width f_G on the housing bore for flanged bushes (Figure 10)

Bore diameter d_G	Chamfer width f_G
$d_G \leq 10$	$1,2 \pm 0,2$
$10 < d_G$	$1,7 \pm 0,2$

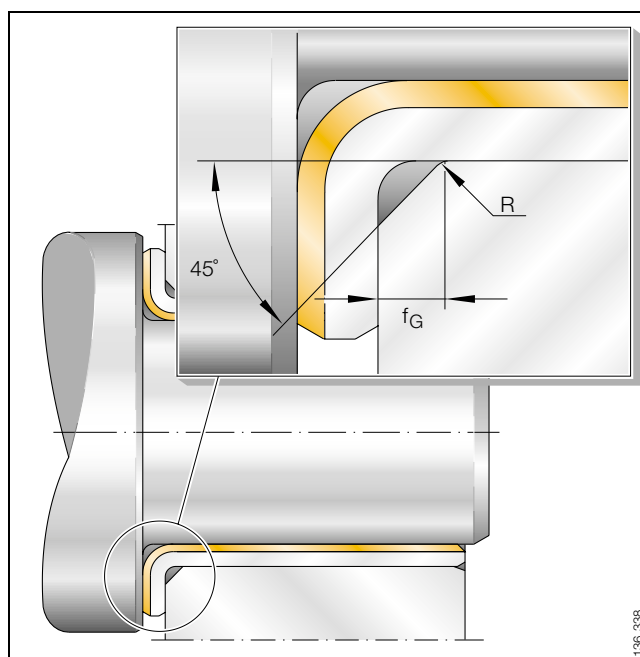


Figure 10 · Chamfer on housing for flanged bush PAF

Location of thrust washers

It is recommended that:

- concentric seating is ensured by means of recesses in the housing (Figure 11)
 - diameters and depths of the recesses: see *dimension tables*
- unintentional rotation is prevented by means of a dowel pin or countersunk head screw (Figures 11 and 12)
 - the screw head or dowel pin must be recessed by at least 0,25 mm with respect to the running surface (Figures 11 and 12)
 - size and arrangement of the holes, see *dimension tables*.

If recesses are not possible in the housing,

- location must be provided by means of several dowel pins or screws (Figure 12)
- other location methods should be used.

It is not always necessary to secure thrust washers against rotation. In some cases, the friction between the bearing backing and the housing will be sufficient.

Strips can be located in the same way as thrust washers.

Other location methods

If the interference fit of the bush is inadequate or if it is uneconomical to use dowel pins or screws, there are economical alternative methods for location:

- laser welding
- soft soldering
- adhesive fastening: see "Further information".

! The temperature of the running-in or sliding surface must not exceed +280 °C for Permaglide® P1 and +140 °C for Permaglide® P2.

The running-in or sliding surface must always be kept free of adhesives.

Recommendation: The adhesive manufacturer should be consulted for information about adhesive fastening, particularly on adhesive selection, preparation of the surface, hardening, strength, temperature range and elongation behaviour.



Further information on adhesive fastening of Permaglide® plain bearing materials:
 INA Technical Product Information
 "Adhesive bonding of Permaglide®, TPI 50"

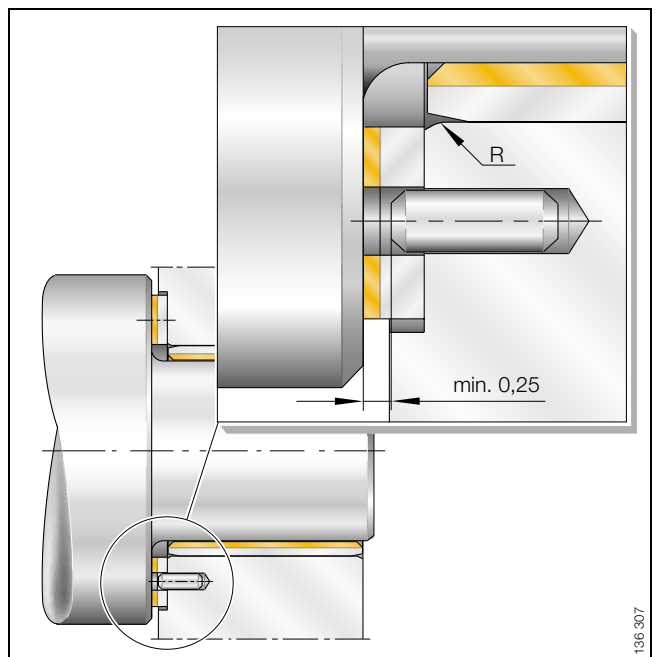


Figure 11 · Location of a PAW thrust washer in a recess in the housing

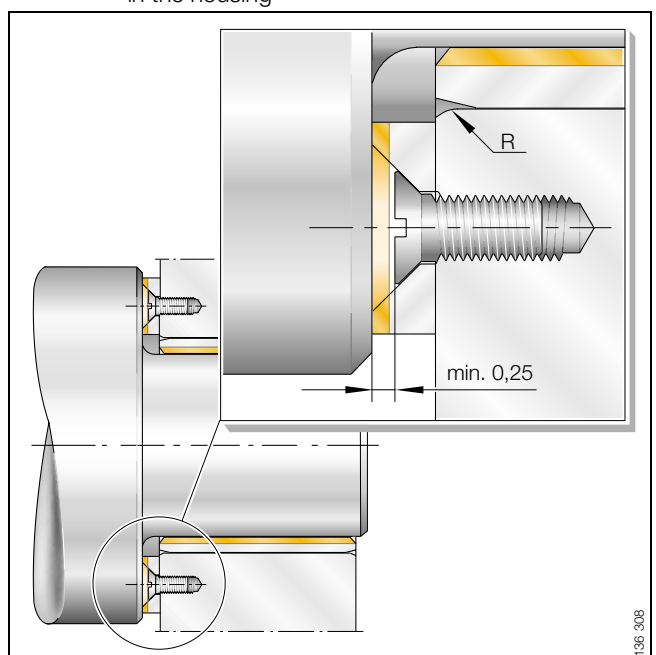


Figure 12 · Location of a PAW thrust washer without a recess in the housing



Shaft


Shafts should be chamfered and all sharp edges rounded:

- this makes fitting easier
- it prevents damage to the sliding layer of the bush.

Mating surface

Optimum operating life

- The optimum operating life will be achieved with a roughness depth of the mating surface of max. R_z2 to R_z3
 - for dry running of Permaglide® P1
 - with lubrication of Permaglide® P2.

 Very small roughness depth values do not have a beneficial effect on the operating life. However, larger roughness depth values do significantly reduce the operating life.

- With Permaglide® P1 and P2, corrosion of the mating surface can be prevented by:
 - sealing
 - the use of corrosion-resistant steel
 - applying a suitable surface treatment.

With Permaglide® P2, the lubricant gives additional protection against corrosion.

- The mating surface should be wider than the bearing to prevent the formation of steps in the sliding layer.

Surface roughness

- Ground or drawn surfaces are preferable
- Even with a surface roughness of R_z2 to R_z3 , precision turned or precision turned and rolled surfaces can cause greater wear due to the spiral grooves produced during precision turning
- Spheroidal graphite cast iron (GGG) has an open surface structure and should therefore be ground to a surface roughness of R_z2 or better.
 - The direction of rotation of cast shafts in the application should be the same as that of the grinding wheel during machining, since increased wear should be anticipated if rotation is in the opposing direction (Figure 13).

Hydrodynamic operation

For hydrodynamic operation, the roughness depth R_z of the mating surface should be less than the smallest lubricant film thickness for fluid friction.

INA offers calculation of hydrodynamic operation as a service.

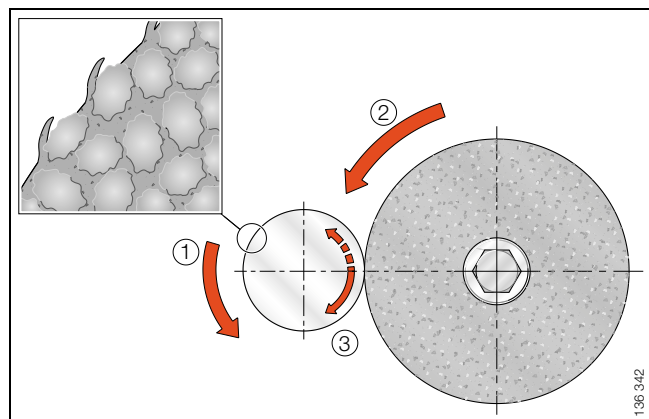


Figure 13 · Grinding of a cast shaft

- ① Direction of shaft rotation in the application
- ② Direction of grinding wheel rotation
- ③ Direction of shaft rotation during grinding

Design of bearing arrangements

Seals

If increased levels of contamination occur or if the bearing is used in an aggressive environment, the bearing arrangement should be protected by (Figure 14):

- the adjacent construction ①
- a gap seal ②
- a shaft seal ③
- a grease collar.

Heat dissipation

Correct and sufficient heat dissipation must be provided.

- If the bearing operates under hydrodynamic conditions, the heat is largely dissipated by the fluid lubricant
- In the case of maintenance-free and low-maintenance plain bearings, the heat can be dissipated through the housing and the shaft.

Machining of bearing elements

- Permaglide® plain bearings can be machined by either cutting or non-cutting methods, such as shortening, drilling or bending
- Permaglide® should be cut starting from the PTFE side, since the cutting process would otherwise leave a burr on the running surface
- The bearing elements must subsequently be cleaned
- Bright steel surfaces (cut edges) must be protected against corrosion by means of:
 - oil or
 - electroplated protective coatings.If high current densities or long coating times are used, the sliding layers should be masked to prevent deposits.



Exceeding the following temperature limits during machining constitutes a health hazard:
+280 °C for Permaglide® P1
+140 °C for Permaglide® P2

The swarf produced may contain lead.

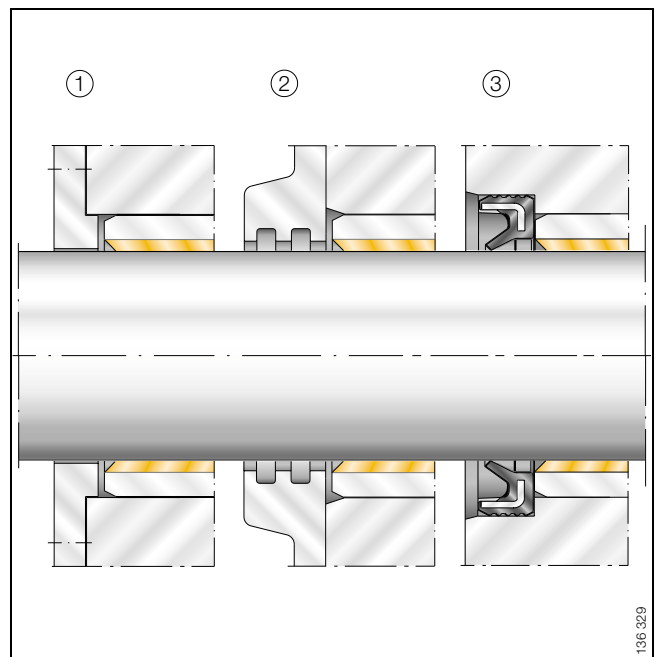


Figure 14 · Seals



Alignment

Accurate alignment is important for radial and axial plain bearings. This applies particularly to dry plain bearings because there is no lubricant film to distribute the load.

The angular misalignment across the entire bush width must not exceed 0,02 mm (Figure 15). This value also applies across the whole width for bushes arranged in pairs as well as to thrust washers.

If the bushes are arranged one behind the other, it may be advisable to use bushes of the same width and the butt joints should be aligned.

High edge loads can be reduced by design measures.

Design measures (Figure 16):

- chamfers
- increased bore diameter in edge area
- bushes projecting beyond the edge of the bore.

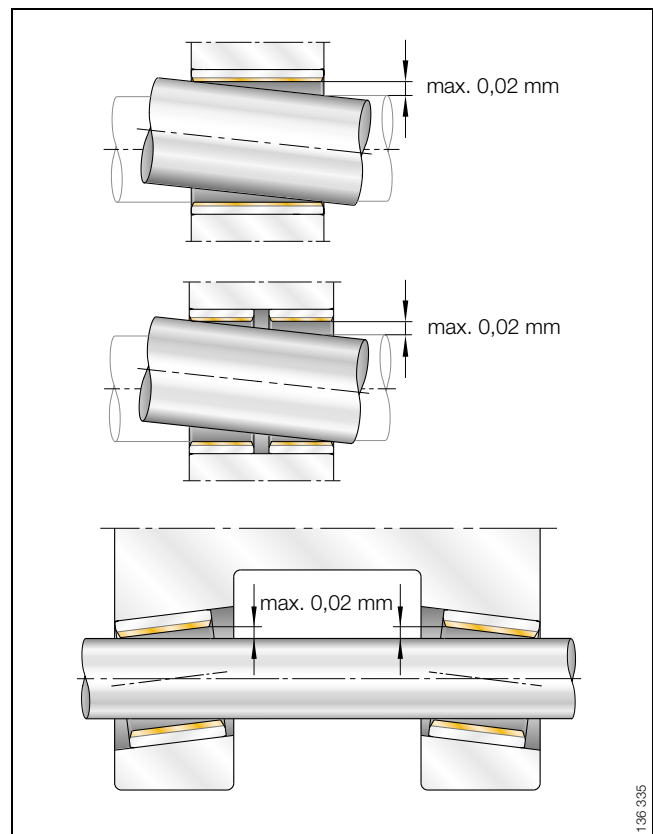


Figure 15 · Permissible angular misalignment

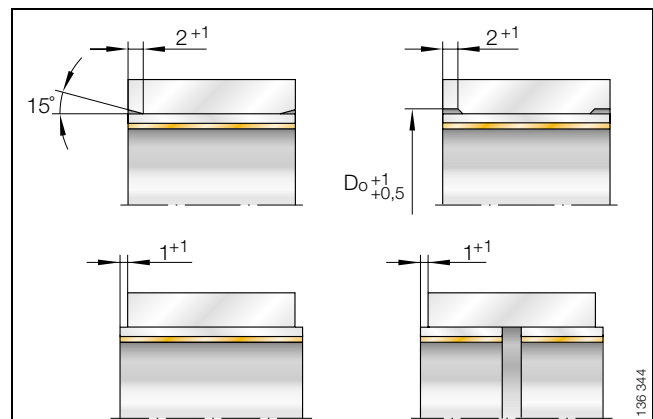


Figure 16 · Reduction of peak stresses at edges

Internal clearance and mounting tolerances

Metric sizes

Theoretical internal clearance

Permaglide® P1 and P2 bushes are pressed into the housing. This provides axial and radial location.

No additional means of location are required.

If the mounting tolerances in Table 7 are used, rigid housings and shafts will have:

- interference fit
- internal clearance according to Table 12, page 26.

The theoretical internal clearance is calculated as follows:

$$\Delta s_{\max} = d_{G\max} - 2 \cdot s_{3\min} - d_{W\min} \quad (11)$$

$$\Delta s_{\min} = d_{G\min} - 2 \cdot s_{3\max} - d_{W\max} \quad (12)$$

Δs_{\max} , Δs_{\min} mm
Maximum internal clearance, minimum internal clearance

$d_{G\max}$, $d_{G\min}$ mm
Maximum housing bore diameter, minimum housing bore diameter

$d_{W\max}$, $d_{W\min}$ mm
Maximum shaft diameter, minimum shaft diameter

$s_{3\max}$, $s_{3\min}$ mm
Maximum wall thickness, minimum wall thickness, see Table 10, page 25.

⚠ Expansion of the housing bore is not taken into account in calculation of the internal clearance.

The interference U is calculated using the tolerances for the housing bore specified in Table 7 and the deviations of the bush outside diameter D_o in Table 8, page 25.

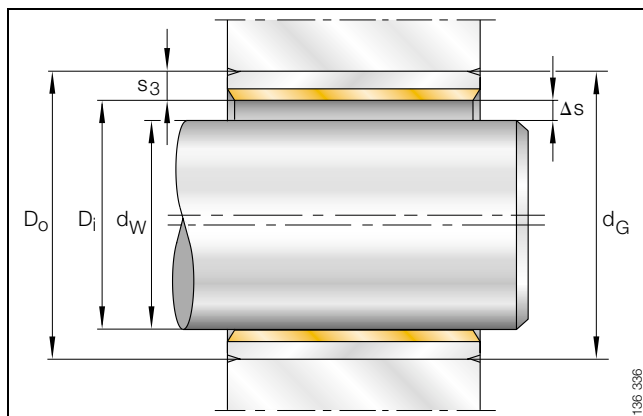


Figure 17 · Theoretical internal clearance Δs

Interference fit and internal clearance

Table 13 on page 27 shows measures that can be taken to influence the internal clearance and interference fit:

- at high ambient temperatures
- depending on housing material
- depending on housing wall thickness.

Reduced tolerances

Reduced clearance tolerances require tighter tolerances for the shaft and the bore.

Table 7 · Recommended mounting tolerances

Diameter range	Permaglide®		
	P10	P11	P20
Shaft			
$d_W < 5$	h6	f7	h8
$5 \leq d_W < 80$	f7	f7	h8
$80 \leq d_W$	h8	h8	h8
Housing bore			
$d_G \leq 5,5$	H6	-	-
$5,5 < d_G$	H7	H7	H7

⚠ If shafts in tolerance zone h are used, the internal clearance must be checked for $5 \leq d_W < 80$ (P10) and $d_W < 80$ (P11) in accordance with the equations (11) for Δs_{\max} and (12) for Δs_{\min} .

Further information

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Conversion to DIN ISO 3547

DIN 1494, which has been applicable up to now, specifies the deviations for outside diameters up to $D_o \leq 180$ mm, and DIN ISO 3547 additionally specifies them for $D_o > 180$ mm.

The deviations for the outside diameters D_o therefore correspond to the specifications in Table 8 or to DIN ISO 3547 for the transitional period.

Deviations for the outside diameter

Table 8 · Deviations for outside diameter D_o

Outside diameter of bush ¹⁾ D_o	Deviations (Test A to DIN ISO 3547-2)			
	P10, P20		P11	
	upper	lower	upper	lower
$D_o \leq 10$	+0,055	+0,025	+0,075	+0,045
$10 < D_o \leq 18$	+0,065	+0,030	+0,080	+0,050
$18 < D_o \leq 30$	+0,075	+0,035	+0,095	+0,055
$30 < D_o \leq 50$	+0,085	+0,045	+0,110	+0,065
$50 < D_o \leq 80$	+0,100	+0,055	+0,125	+0,075
$80 < D_o \leq 120$	+0,120	+0,070	+0,140	+0,090
$120 < D_o \leq 180$	+0,170	+0,100	+0,190	+0,120
$180 < D_o \leq 250$	+0,210	+0,130	+0,230	+0,150
$250 < D_o \leq 305$	+0,260	+0,170	+0,280	+0,190

¹⁾ Deviations to DIN ISO 3547-1, Table 6.

Wall thicknesses with tolerances

Table 9 · Wall thickness s_3 for bushes and flanged bushes P1

Inside diameter D_i	Wall thickness s_3	Deviations to DIN ISO 3547-1, Table 3, series B			
		P10 ¹⁾		P11	
		upper	lower	upper	lower
$D_i < 5$	0,75	0	-0,020	-	-
	1	-	-	+0,005	-0,020
$5 \leq D_i < 20$	1	+0,005	-0,020	+0,005	-0,020
$20 \leq D_i < 28$	1,5	+0,005	-0,025	+0,005	-0,025
$28 \leq D_i < 45$	2	+0,005	-0,030	+0,005	-0,030
$45 \leq D_i < 80$	2,5	+0,005	-0,040	+0,005	-0,040
$80 \leq D_i < 120$	2,5	-0,010	-0,060	-0,010	-0,060
$120 \leq D_i$	2,5	-0,035	-0,085	-0,035	-0,085

¹⁾ Permaglide® P10 is also available with a wall thickness of 0,5 mm; please consult the INA engineering service.

Table 10 · Wall thickness s_3 for Permaglide® P20 bushes

Inside diameter D_i	Wall thickness s_3	Deviations to DIN ISO 3547-1, Table 3, series D, P20	
		upper	lower
$8 \leq D_i < 20$	1	-0,020	-0,045
$20 \leq D_i < 28$	1,5	-0,025	-0,055
$28 \leq D_i < 45$	2	-0,030	-0,065
$45 \leq D_i < 80$	2,5	-0,040	-0,085
$80 \leq D_i$	2,5	-0,050	-0,115

Chamfers and chamfer tolerances

Table 11 · Outer chamfer C_o and inner edge break C_i (Figure 18) for bushes in metric sizes, to DIN ISO 3547-1, Table 2

Wall thickness s_3	Outer chamfer, machined without cutting ¹⁾ C_o	Inner edge break C_i	
		min.	max.
0,75	$0,5 \pm 0,3$	0,1	0,4
1	$0,6 \pm 0,4$	0,1	0,5
1,5	$0,6 \pm 0,4$	0,1	0,7
2	$1,0 \pm 0,4$	0,1	0,7
2,5	$1,2 \pm 0,4$	0,2	1,0

C_o should be correspondingly larger for bushes with a bearing bore that must be machined to size. Chamfer deformation due to round bending is permissible.

¹⁾ For a transitional period, bushes with differing chamfers can be supplied for wall thicknesses 2 mm and 2,5 mm.

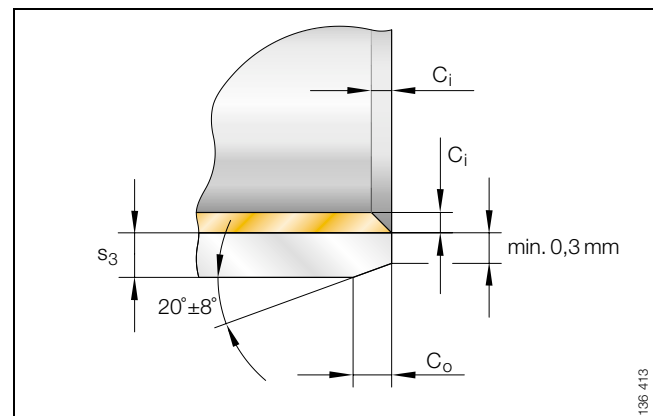


Figure 18 · Outer chamfer C_o and inner edge break C_i for metric sizes

Internal clearance and mounting tolerances

Metric sizes

Table 12 · Theoretical internal clearance after pressing in of bushes or flanged bushes of metric sizes, disregarding possible expansion of the bore. Equations: page 24

Bush diameter		Internal clearance Δs			
		P10, P11		P20	
D_i mm	D_o mm	Δs_{\min} mm	Δs_{\max} mm	Δs_{\min} mm	Δs_{\max} mm
2	3,5	0	0,054	–	–
3	4,5	0	0,054	–	–
4	5,5	0	0,056	–	–
5	7	0	0,077	–	–
6	8	0	0,077	–	–
7	9	0,003	0,083	–	–
8	10	0,003	0,083	0,040	0,127
10	12	0,003	0,086	0,040	0,130
12	14	0,006	0,092	0,040	0,135
13	15	0,006	0,092	–	–
14	16	0,006	0,092	0,040	0,135
15	17	0,006	0,092	0,040	0,135
16	18	0,006	0,092	0,040	0,135
18	20	0,006	0,095	0,040	0,138
20	23	0,010	0,112	0,050	0,164
22	25	0,010	0,112	0,050	0,164
24	27	0,010	0,112	0,050	0,164
25	28	0,010	0,112	0,050	0,164
28	32	0,010	0,126	0,060	0,188
30	34	0,010	0,126	0,060	0,188
32	36	0,015	0,135	0,060	0,194
35	39	0,015	0,135	0,060	0,194
40	44	0,015	0,135	0,060	0,194
45	50	0,015	0,155	0,080	0,234
50	55	0,015	0,160	0,080	0,239
55	60	0,020	0,170	0,080	0,246
60	65	0,020	0,170	0,080	0,246
65	70	0,020	0,170	–	–
70	75	0,020	0,170	0,080	0,246
75	80	0,020	0,170	0,080	0,246
80	85	0,020	0,201	0,100	0,311
85	90	0,020	0,209	–	–

Bush diameter		Internal clearance Δs			
		P10, P11		P20	
D_i mm	D_o mm	Δs_{\min} mm	Δs_{\max} mm	Δs_{\min} mm	Δs_{\max} mm
90	95	0,020	0,209	0,100	0,319
95	100	0,020	0,209	–	–
100	105	0,020	0,209	0,100	0,319
105	110	0,020	0,209	–	–
110	115	0,020	0,209	–	–
115	120	0,020	0,209	–	–
120	125	0,070	0,264	–	–
125	130	0,070	0,273	–	–
130	135	0,070	0,273	–	–
135	140	0,070	0,273	–	–
140	145	0,070	0,273	–	–
150	155	0,070	0,273	–	–
160	165	0,070	0,273	–	–
180	185	0,070	0,279	–	–
200	205	0,070	0,288	–	–
220	225	0,070	0,288	–	–
250	255	0,070	0,294	–	–
300	305	0,070	0,303	–	–

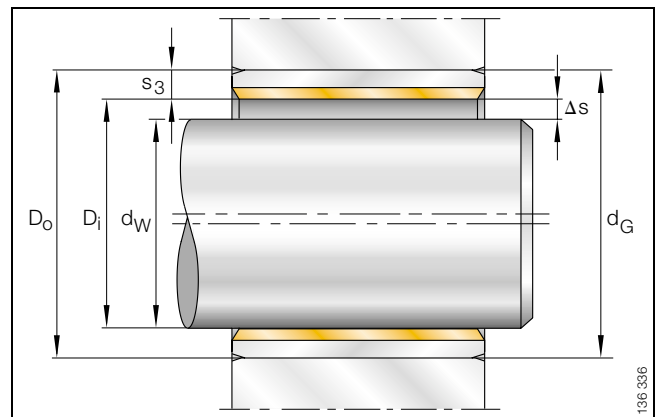


Figure 19 · Theoretical internal clearance Δs



Table 13 · Consequences and measures for interference fit and internal clearance in case of high ambient temperatures, special housing materials or special wall thicknesses

Design and environmental influences	Consequence	Measure	Note
Light metal or thin-walled housings	Large expansion, excessive clearance	Reduce housing bore d_G	Housing is more highly stressed; the permissible stress on the housing must not be exceeded.
Steel or cast iron housings <i>at high ambient temperatures</i>	Reduced clearance	Reduce shaft diameter d_W by 0,008 mm for each increment of 100 °C above room temperature	
Bronze or copper alloy housings <i>at high ambient temperatures</i>	Impaired interference fit	Reduce housing bore d_G ; recommended diameter change for each increment of 100 °C above room temperature: $d_G - 0,05\%$	Reduce shaft diameter d_W by the same value to maintain the internal clearance.
Aluminium alloy housings <i>at high ambient temperatures</i>	Impaired interference fit	Reduce housing bore d_G ; recommended diameter change for each increment of 100 °C above room temperature: $d_G - 0,1\%$	Reduce shaft diameter d_W by the same value to maintain the internal clearance. The housing is more highly stressed at temperatures below 0 °C; the permissible stress on the housing must not be exceeded.
Bushes with thicker corrosion protection layer	Outer diameter D_o too large, insufficient clearance	Increase housing bore d_G Example: Layer thickness $0,015 \pm 0,003$ mm Consequently: $d_G + 0,03$ mm	Bush and housing will be more highly stressed if appropriate measures are not taken.

Internal clearance and mounting tolerances

Inch sizes

Table 14 · Theoretical internal clearance (Figure 20, page 29) after pressing in of PAPZ bushes – disregarding possible expansion of the housing bore – and shaft and bore dimensions

Shaft diameter	Designation	Bush diameter Inch/mm		Wall thickness Inch/mm		Shaft dimensions Inch/mm		Bore dimensions Inch/mm		Internal clearance Inch/mm	
		D _i	D _o	S ₃ min	S ₃ max	d _W min	d _W max	d _G min	d _G max	Δs _{min}	Δs _{max}
3/16	PAPZ 03	0,1875	0,25	0,0307	0,0315	0,1858	0,1865	0,2497	0,2503	0,0002	0,0031
		4,763	6,35	0,780	0,800	4,719	4,737	6,342	6,358	0,005	0,079
1/4	PAPZ 04	0,25	0,3125	0,0307	0,0315	0,2481	0,2490	0,3122	0,3128	0,0002	0,0033
		6,35	7,938	0,780	0,800	6,302	6,325	7,930	7,945	0,005	0,083
5/16	PAPZ 05	0,3125	0,375	0,0307	0,0315	0,3106	0,3115	0,3747	0,3753	0,0002	0,0033
		7,938	9,525	0,780	0,800	7,889	7,912	9,517	9,533	0,005	0,083
3/8	PAPZ 06	0,375	0,4688	0,0461	0,0471	0,3731	0,3740	0,4684	0,4691	0,0002	0,0038
		9,525	11,906	1,171	1,196	9,477	9,500	11,897	11,915	0,005	0,096
7/16	PAPZ 07	0,4375	0,5312	0,0461	0,0471	0,4355	0,4365	0,5309	0,5316	0,0002	0,0039
		11,113	13,494	1,171	1,196	11,062	11,087	13,485	13,503	0,006	0,099
1/2	PAPZ 08	0,5	0,5938	0,0461	0,0471	0,4980	0,4990	0,5934	0,5941	0,0002	0,0039
		12,7	15,081	1,171	1,196	12,649	12,675	15,072	15,090	0,005	0,099
9/16	PAPZ 09	0,5625	0,6563	0,0461	0,0471	0,5605	0,5615	0,6559	0,6566	0,0002	0,0039
		14,288	16,669	1,171	1,196	14,237	14,262	16,660	16,678	0,006	0,099
5/8	PAPZ 10	0,625	0,7188	0,0461	0,0471	0,6230	0,6240	0,7184	0,7192	0,0002	0,0040
		15,875	18,256	1,171	1,196	15,824	15,850	18,247	18,268	0,005	0,102
1 1/16	PAPZ 11	0,6875	0,7813	0,0461	0,0471	0,6855	0,6865	0,7809	0,7817	0,0002	0,0040
		17,463	19,844	1,171	1,196	17,412	17,437	19,835	19,855	0,006	0,101
3/4	PAPZ 12	0,75	0,875	0,0615	0,0627	0,7479	0,7491	0,8747	0,8755	0,0002	0,0046
		19,050	22,225	1,562	1,593	18,997	19,027	22,217	22,238	0,004	0,117
7/8	PAPZ 14	0,875	1	0,0615	0,0627	0,8729	0,8741	0,9997	1,0005	0,0002	0,0046
		22,225	25,4	1,562	1,593	22,172	22,202	25,392	25,413	0,004	0,117
1	PAPZ 16	1	1,125	0,0615	0,0627	0,9979	0,9991	1,1247	1,1255	0,0002	0,0046
		25,4	28,575	1,562	1,593	25,347	25,377	28,567	28,588	0,004	0,117
1 1/8	PAPZ 18	1,125	1,2813	0,0770	0,0784	1,1226	1,1238	1,2808	1,2818	0,0002	0,0052
		28,575	32,544	1,956	1,991	28,514	28,545	32,532	32,558	0,005	0,132
1 1/4	PAPZ 20	1,25	1,4063	0,0770	0,0784	1,2472	1,2488	1,4058	1,4068	0,0002	0,0056
		31,75	35,719	1,956	1,991	31,679	31,720	35,707	35,733	0,005	0,142
1 3/8	PAPZ 22	1,375	1,5313	0,0770	0,0784	1,3722	1,3738	1,5308	1,5318	0,0002	0,0056
		34,925	38,894	1,956	1,991	34,854	34,895	38,882	38,908	0,005	0,142
1 1/2	PAPZ 24	1,5	1,6563	0,0770	0,0784	1,4972	1,4988	1,6558	1,6568	0,0002	0,0056
		38,1	42,069	1,956	1,991	38,029	38,070	42,057	42,083	0,005	0,142
1 5/8	PAPZ 26	1,625	1,7813	0,0770	0,0784	1,6222	1,6238	1,7808	1,7818	0,0002	0,0056
		41,275	45,244	1,956	1,991	41,204	41,245	45,232	45,258	0,005	0,142
1 3/4	PAPZ 28	1,75	1,9375	0,0923	0,0941	1,7471	1,7487	1,9371	1,9381	0,0002	0,0065
		44,45	49,213	2,344	2,390	44,376	44,417	49,202	49,228	0,005	0,164
2	PAPZ 32	2	2,1875	0,0923	0,0941	1,9969	1,9987	2,1871	2,1883	0,0002	0,0069
		50,8	55,563	2,344	2,390	50,721	50,767	55,552	55,583	0,005	0,174



Table 15 · Outer chamfer C_o and inner edge break C_i (Figure 21) for bushes in inch sizes

	Wall thickness <i>Inch/mm</i>		Outer chamfer <i>Inch/mm</i>		Inner edge break <i>Inch/mm</i>	
	s_3 min.	s_3 max.	C_o min.	C_o max.	C_i min.	C_i max.
<i>Inch</i>	0,0307	0,0315	0,008	0,032	0,0040	0,016
mm	0,780	0,800	0,2	0,8	0,1	0,4
<i>Inch</i>	0,0461	0,0471	0,008	0,040	0,0040	0,020
mm	1,171	1,196	0,2	1,0	0,1	0,5
<i>Inch</i>	0,0615	0,0627	0,008	0,040	0,0040	0,028
mm	1,562	1,593	0,2	1,0	0,1	0,7
<i>Inch</i>	0,0770	0,0784	0,024	0,055	0,0040	0,028
mm	1,956	1,991	0,6	1,4	0,1	0,7
<i>Inch</i>	0,0923	0,0941	0,032	0,063	0,0080	0,040
mm	2,344	2,390	0,8	1,6	0,2	1,0

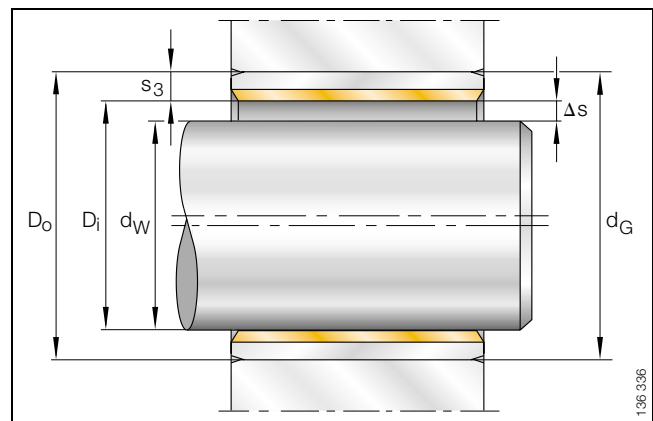


Figure 20 · Theoretical internal clearance Δs

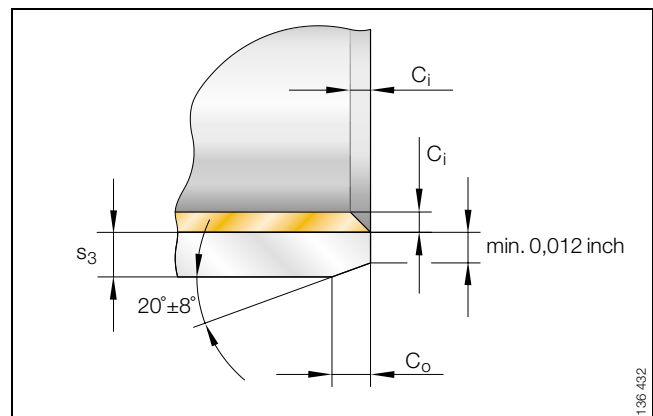


Figure 21 · Outer chamfer C_o and inner edge break C_i for inch sizes

Pressing in of bushes

Permaglide® bushes can be easily pressed into the housing bore

- The bushes can be pressed in more easily if the bush backing or housing bore is lightly oiled.

Recommendations

Outside diameter D_o up to approx. 55 mm:

- press in flush using arbor without auxiliary ring according to Figure 22
- press in recessed using arbor without auxiliary ring according to Figure 23.

Outside diameter greater than approx. 55 mm:

- press in using arbor with auxiliary ring according to Figure 24, page 31.



- Ensure that assembly is carried out in clean conditions. Contamination will reduce the operating life of the bearing arrangement.
- Ensure that the sliding layer is not damaged.
- If a mounting position is specified, this must be observed.

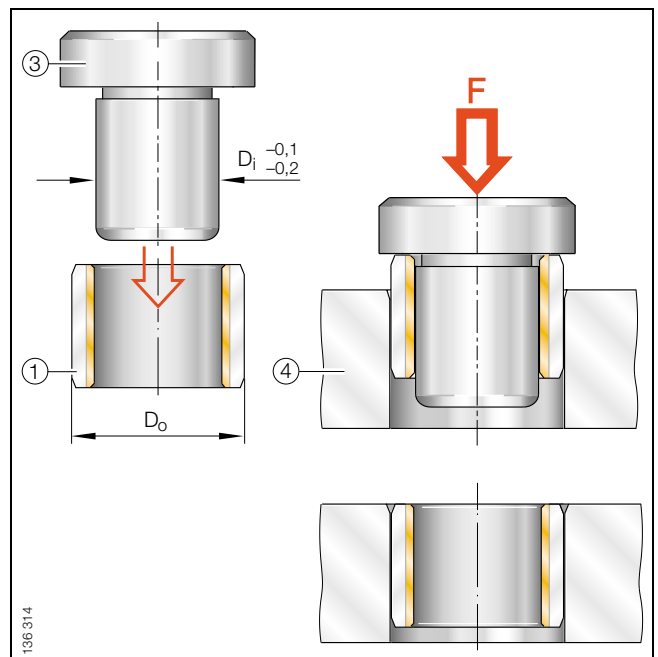


Figure 22 · Pressing in flush, $D_o \leq 55$ mm
Legend: see page 31

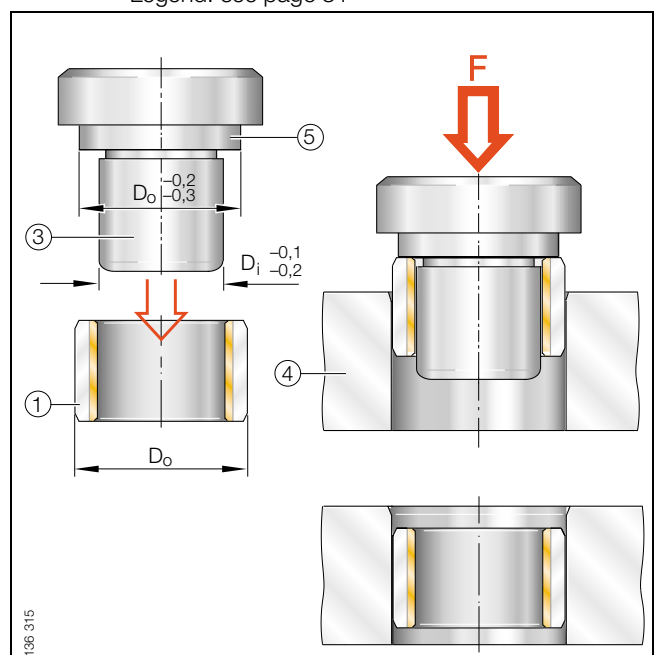


Figure 23 · Pressing in recessed, $D_o \geq 55$ mm
Legend: see page 31

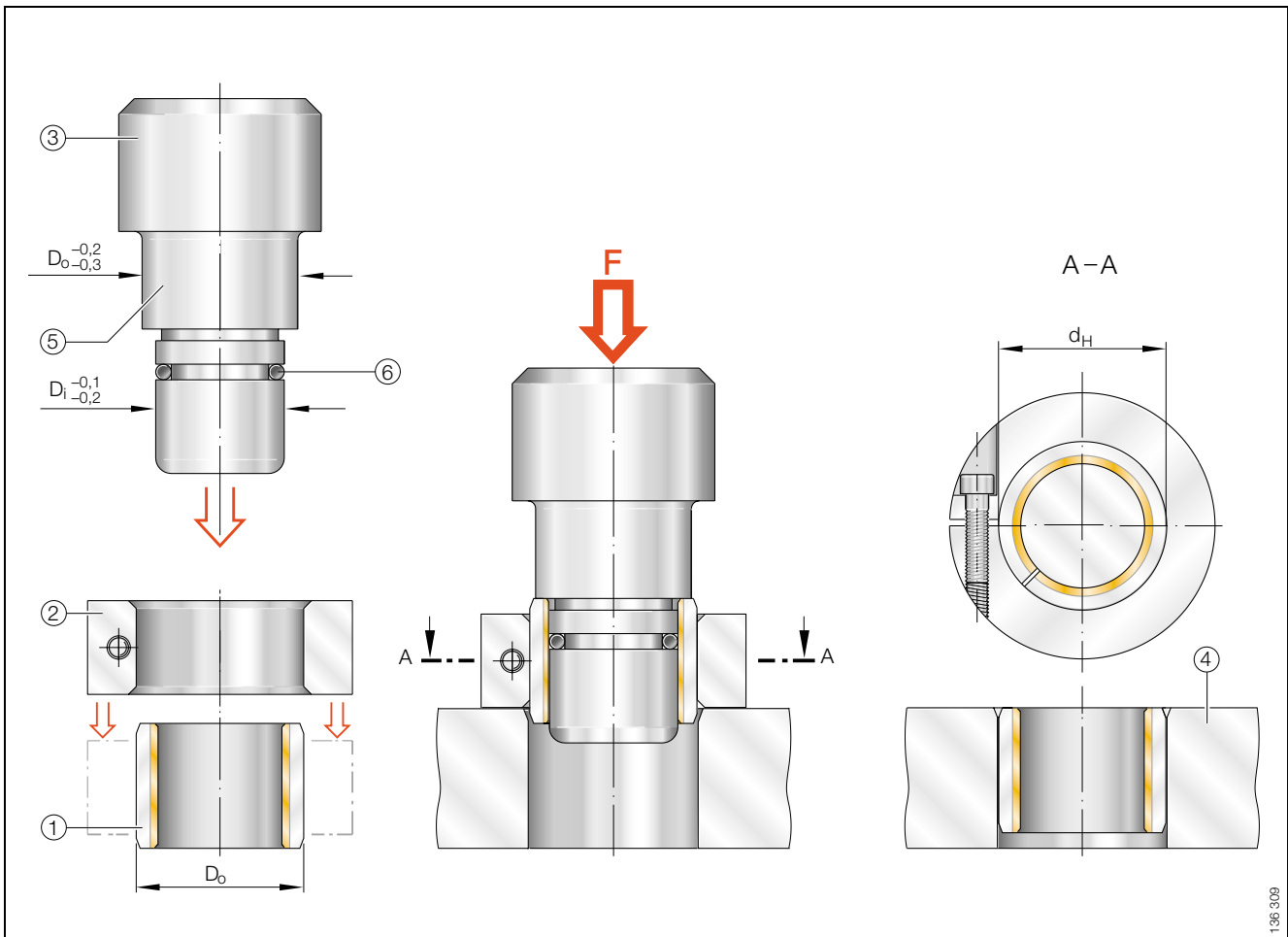


Figure 24 · Pressing in of bushes, $D_o \geq 55$ mm, with auxiliary ring

- ① Bush
- ② Auxiliary ring
- ③ Press-in arbor
- ④ Housing
- ⑤ Shoulder diameter
- ⑥ O ring

Table 16 · Inside diameter d_H of auxiliary ring for outside diameter D_o

D_o mm	d_H mm
$55 \leq D_o \leq 100$	$D_o^{+0,28}_{+0,25}$
$100 < D_o \leq 200$	$D_o^{+0,40}_{+0,36}$
$200 < D_o \leq 305$	$D_o^{+0,50}_{+0,46}$

Calculation of press-in force

- The press-in force is dependent on many factors that are difficult to assess accurately, e.g.:
 - actual interference
 - coefficient of friction
 - scoring
 - press-in speed.
- Simplifications are therefore permissible in the calculation, e.g. for the interference:
 - only the thickness of the steel or bronze backing s_1 is taken into account.



The calculated press-in force is only a guide value.

Considerably higher forces sometimes occur in practice! In critical cases, the press-in force must be determined by means of tests.

INA offers calculation of the press-in force as a service.

Table 17 · Modulus of elasticity, transversal contraction

Material	Modulus of elasticity E N/mm ²	Transversal contraction ν
Steel	210 000	0,30
Cast steel	210 000	0,30
GGG	170 000	0,28
Aluminium	70 000	0,33
Bronze	85 000	0,35

Table 18 · Guide values for the coefficients of friction μ_R between bush backing and housing bore

Housing material	Coefficient of friction μ_L			
	Steel backing, tin plated		Bronze backing	
	dry	lubricated	dry	lubricated
Steel	0,12	0,10	0,10	0,08
Cast steel, GGG	0,12	0,10	0,10	0,08
Aluminium	0,10	0,08	0,10 ¹⁾	0,08 ¹⁾

1)



Caution: Electrochemical contact corrosion!

Calculation steps for determining the press-in force

Press-in force:

$$F = p_1 \cdot \mu_L \cdot A \quad (13)$$

Joint pressure:

$$p_1 = \frac{U}{d_G} \cdot \frac{E_G}{KG + \frac{E_G}{E_L} \cdot KL} \quad (14)$$

$$p_1 = \frac{\frac{U}{d_G} \cdot E_G}{\left[\frac{\left(1 + 2 \frac{s_G}{d_G}\right)^2 + 1}{\left(1 + 2 \frac{s_G}{d_G}\right)^2 - 1} + \nu_G \right] + \frac{E_G}{E_L} \left[\frac{1 + \left(1 - 2 \frac{s_1}{D_o}\right)^2}{1 - \left(1 - 2 \frac{s_1}{D_o}\right)^2} - \nu_L \right]} \quad (15)$$

Outside surface area of bush:

$$A = D_o \cdot \pi \cdot B \quad (16)$$



Calculation of individual expressions

Interference for F_{\max}

$$U_{\max} = D_{o\max} - d_{G\min} - 0,8 \cdot (R_{zG} + R_{zL}) \quad (17)$$

Interference for F_{\min}

$$U_{\min} = D_{o\min} - d_{G\max} - 0,8 \cdot (R_{zG} + R_{zL}) \quad (18)$$

Thickness of steel or bronze backing

$$s_1 = s_3 - 0,3 \text{ mm} \quad (19)$$

“Round brackets”

$$\left(1 + 2 \frac{s_G}{d_G}\right) = RG \quad (20)$$

$$\left(1 - 2 \frac{s_1}{D_o}\right) = RL \quad (21)$$

“Square brackets”

$$\left[\frac{RG^2 + 1}{RG^2 - 1} + \nu_G\right] = KG \quad (22)$$

$$\left[\frac{1 + RL^2}{1 - RL^2} - \nu_L\right] = KL \quad (23)$$

A mm^2

Outside surface area

B mm

Width of bush, see *dimension tables*

D_o mm

Outside diameter of bush (nominal dimension), see *dimension tables*

$d_G = D_o$ mm

Diameter of housing bore (nominal dimension)

$d_{W\max}, d_{W\min}$ mm

Minimum and maximum dimensions of bush; see page 25 for deviations

$d_{G\min}, d_{G\max}$ mm

Minimum and maximum dimensions of bush; see page 24 for tolerances

E_L, E_G N/mm^2

Modulus of elasticity, Table 17, page 32

F N

Press-in force

p_1 N/mm^2

Joint pressure

R_{zG} mm

Roughness depth of housing bore

R_{zL} mm

Roughness depth of bush backing

s_1 mm

Thickness of steel or bronze backing

s_3 mm

Wall thickness of bush, Table 9, page 25

s_G mm

Wall thickness of housing

U mm

Interference

μ_L $-$

Coefficient of friction, Table 18, page 32

ν_L, ν_G $-$

Transversal contraction, Table 17, page 32.

Calculation example

Bush PAP 2010 P10 (steel backing)
Aluminium housing, wall thickness 30 mm
pressed in dry

■ Required:

– press-in force F_{\max} in kN

■ Given:

Outside diameter of bush	D_o	23	mm
Width of bush	B	10	mm
Diameter of housing bore	d_G	23	mm
Wall thickness of housing	s_G	30	mm
Roughness depth of bush backing	R_{zL}	0,006	mm
Minimum dimension of housing bore	$d_{G\min}$	23,000	mm
Maximum dimension of bush	$D_{o\max}$	23,075	mm
Wall thickness of bush	s_3	1,5	mm
Modulus of elasticity of bearing	E_L	210 000	N/mm ²
Modulus of elasticity of housing	E_G	70 000	N/mm ²
Transversal contraction of bearing	ν_L	0,30	
Transversal contraction of housing	ν_G	0,33	
Roughness depth of housing bore	R_{zG}	0,010	mm
Coefficient of friction	μ_R	0,10	

Calculation according to steps on page 32

Interference for F_{\max}

$$U_{\max} = 23,075 \text{ mm} - 23,0 \text{ mm} - 0,8 \cdot (0,010 + 0,006) \text{ mm}$$

$$U_{\max} = 0,062 \text{ mm}$$

Thickness of steel backing

$$s_1 = 1,5 \text{ mm} - 0,3 \text{ mm} = 1,2 \text{ mm}$$

“Round brackets”

$$\left(1 + 2 \cdot \frac{30 \text{ mm}}{23 \text{ mm}}\right) = RG = 3,609$$

$$\left(1 - 2 \cdot \frac{1,2 \text{ mm}}{23 \text{ mm}}\right) = RL = 0,896$$

“Square brackets”

$$\left[\frac{3,609^2 + 1}{3,609^2 - 1} + 0,33\right] = KG = 1,496$$

$$\left[\frac{1 + 0,896^2}{1 - 0,896^2} - 0,30\right] = KL = 8,843$$

Joint pressure

$$p_{1\max} = \frac{0,062}{23} \cdot \frac{70\,000}{1,496 + \frac{70\,000}{210\,000} \cdot 8,843} \text{ N/mm}^2$$

$$p_{1\max} = 42,5 \text{ N/mm}^2$$

Outside surface area of bush

$$A = 23 \text{ mm} \cdot \pi \cdot 10 \text{ mm} = 723 \text{ mm}^2$$

Press-in force

$$F_{\max} = 42,5 \text{ N/mm}^2 \cdot 0,10 \cdot 723 \text{ mm}^2$$

$$F_{\max} = 3073 \text{ N}$$

$$F_{\max} \approx 3 \text{ kN}$$

Delivered condition Storage

Environmental considerations Health and safety at work



Delivered condition

- packed in cartons or
- packed in bags in cartons.

Storage

Permaglide® plain bearings should be stored in:

- clean, dry areas
- at as constant a temperature as possible
- at a relative humidity of 65%.



Keep packaging closed as far as possible.

Permaglide® plain bearings should not be removed from their original packaging until immediately before fitting.

Environmental considerations, health and safety at work

In the interests of the user, the applicable legal regulations and other directives relating to

- environmental protection
- health and safety at work

and similar issues should be observed.

Materials

Permaglide® P1
Maintenance-free plain bearing material



Features 36



Design and safety information 38



Special designs..... 43

Page



Features

Permaglide® plain bearing material P1

- maintenance-free
- suitable for
 - dry running
 - rotary movement
 - oscillating movement and
 - linear movement with short stroke lengths (page 16)
- good sliding characteristics – no stick-slip
- low coefficient of friction
- low wear
- high chemical resistance
- does not tend to weld to metal
- largely resistant to swelling
 - see information on chemical resistance (page 38)
 - does not absorb water
- hydrodynamic operation possible.

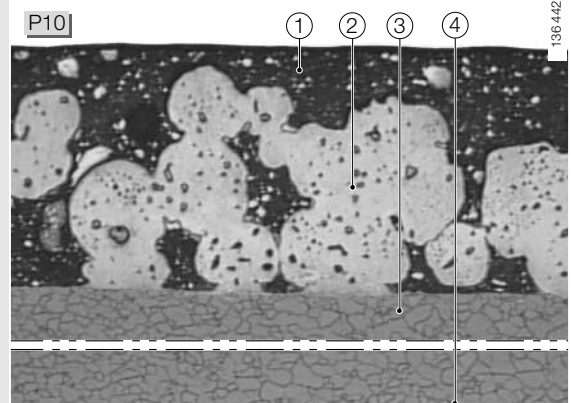
Variants

- P10 with steel backing
- P11 with bronze backing, therefore
 - largely corrosion-resistant
 - very good thermal conductivity
 - antimagnetic.



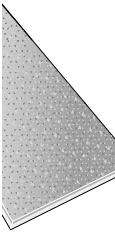
Permaglide® P10 and P11 contain lead (Pb). Therefore, contact with foodstuffs or pharmaceutical products should be avoided.

Permaglide® plain bearing material, maintenance-free

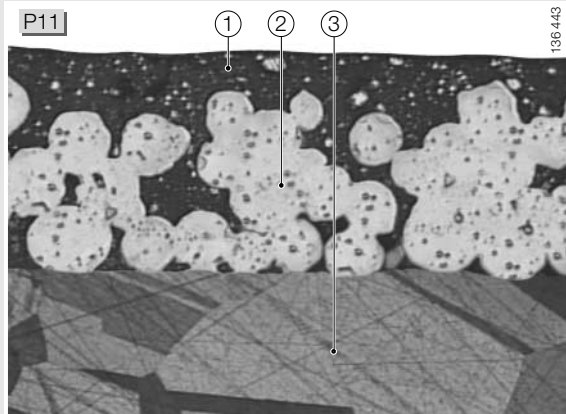


- Running-in layer ①: polytetrafluoroethylene (PTFE) and lead (Pb), 0,01 mm to 0,03 mm thick
- Sliding layer ②: porous bronze layer filled with PTFE/Pb, 0,20 mm to 0,35 mm thick
- Steel backing ③
- Surface protection for steel backing, end faces and butt joint faces ④: tin, approx. 0,002 mm thick

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Permaglide® plain bearing material, maintenance-free



- Running-in layer ①: polytetrafluoroethylene (PTFE) and lead (Pb), 0,01 mm to 0,03 mm thick
- Sliding layer ②: porous bronze layer filled with PTFE/Pb, 0,20 mm to 0,35 mm thick
- Bronze backing ③

Technical data

Maximum pv value for dry running	continuous operation	pv	1,8	N/mm ² · m/s
	for short periods	pv	3,6	N/mm ² · m/s
Permissible specific bearing load	static	ρ_{\max}	250	N/mm ²
	very low sliding speed	ρ_{\max}	140	N/mm ²
	rotating, oscillating	ρ_{\max}	56	N/mm ²
Permissible sliding speed	dry running	v_{\max}	2	m/s
	hydrodynamic operation	v_{\max}	> 2	m/s
Permissible operating temperature	–	ϑ	– 200 to +280	°C
Coefficient of thermal expansion	steel backing	α_{St}	$11 \cdot 10^{-6}$	K ⁻¹
	bronze backing	α_{Bz}	$17 \cdot 10^{-6}$	K ⁻¹
Coefficient of thermal conductivity	steel backing	λ_{St}	> 42	W (m · K) ⁻¹
	bronze backing	λ_{Bz}	> 70	W (m · K) ⁻¹
Relative electrical resistance after running-in		$R_{\text{rel min}}$	> 1	$\Omega \cdot \text{cm}^2$



Design and safety information

Friction

Sliding movements are free from stick-slip.

The friction is dependent on:

- the roughness depth of the mating surface
- the mating surface material
- the specific bearing load
- the sliding speed
- the operating temperature
 - up to approx. +100 °C, the coefficient of friction is slightly less than the value at room temperature
 - over +100 °C, the coefficient of friction can be up to 50% higher than the value at room temperature.

The coefficient of friction is more favourable at high specific bearing load and low sliding speed (Table 19).

Table 19 · Coefficient of friction at room temperature (steel mating surface, roughness depth R_{z2} to R_{z3})

Specific bearing load p N/mm ²		Sliding speed v m/s		Coefficient of friction μ	
250 to 140	high	to 0,001	low	0,03	low
140 to 60	↑ p ↑	0,001 to 0,005	v ↓	0,04 to 0,07	μ ↓
60 to 10		0,005 to 0,05		0,07 to 0,1	
10 to 1		0,05 to 0,5		0,1 to 0,15	
to 1		0,5 to 2		0,15 to 0,25	
	low		high		

Chemical resistance and corrosion protection

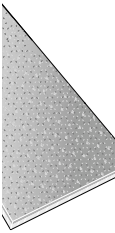
The resistance of Permaglide® P1 depends on the chemical characteristics of the individual layers.

In general:

- Permaglide® P1 is resistant to water, alcohols, glycols and many mineral oils.
- With Permaglide® P10, the tin-plated steel surface is adequate protection against corrosion in most cases – special designs (page 43).
- With Permaglide® P11, the bronze backing is additionally resistant to steam, seawater and various saline solutions.

Special operating conditions

- The running-in layer and sliding layer swells in the presence of some mineral oils at temperatures above +100 °C.
Remedy:
 - increase the internal clearance
 - use special design Permaglide® P14.
- Permaglide® P10 is not resistant to acidic (pH <5) and alkaline media (pH >9).
- The bronze backing of Permaglide® P11 is not resistant to oxidizing acids and gases such as halides, ammonia or hydrogen sulphide, especially if these gases have a high moisture content.



Fretting corrosion

Permaglide® P1 is designed for dry running, i.e. there is no lubricant to protect the mating surface.

If there is a risk of corrosion on the mating surface, the following materials are advisable:

- corrosion-resistant steels
- hard-chromium plated steels
- hard-anodized aluminium.

Corrosion-resistant surfaces of this type also decrease the wear.

Rust cannot form between the sliding material and mating surface due to the running-in layer or sliding surface comprising PTFE and Pb.

Fretting corrosion occurs only infrequently between the steel backing of Permaglide® P10 and the housing. Electroplated protective layers delay corrosion in such cases.

Electrochemical contact corrosion

In unfavourable conditions, local elements can form which reduce the operating life.

Remedy:

- note this factor at the design stage
- clarify by means of tests if necessary.

Electrical conductivity

The electrical conductivity of new bearings may be lower because the running-in layer is still present. The bronze layer is partially exposed after the running-in process, thus improving the electrical conductivity.

The electrical resistance is dependent on the size of the contact surface area.

Lubrication

Permaglide® P1 contains dry lubricants and therefore does not need lubrication.

Permaglide® P1 can be used in fluid media in certain applications. The improved heat dissipation in this case can significantly increase the operating life.

The compatibility of the media with Permaglide® P1 must be checked.



One-off lubrication is not permissible.

Oil and grease lubrication, even in very small quantities, impairs material transfer during running-in.

Paste formation

Over time, grease mixes with abraded particles from the bearing to form a paste that promotes wear. Solid lubricants such as zinc sulphide, molybdenum disulphide or similar grease additives promote this paste formation. For this reason, they are not permissible.

Exceptions

If grease lubrication is unavoidable in exceptional cases, the bearing should be relubricated regularly in order to counteract paste formation.

Examples:

- corrosion protection of the mating surface
- simple sealing against contamination.

However, a corrosion-protected mating surface (page 21) or a different seal type for the bearing (page 22) is nevertheless more advantageous in such cases.

Hydrodynamic properties

Permaglide® plain bearings P1 can be operated under hydrodynamic conditions.

Advantages:

- higher circumferential speeds permissible than with dry running
- wear-free operation, since only fluid friction is present once the transition speed is reached
- self-lubricating action of Permaglide® P1 in mixed friction conditions (below the transition speed).

INA offers the calculation of hydrodynamic operating conditions for Permaglide® bearings as a service.

The following data must be provided:

- load
- speed
- housing bore diameter d_G with tolerance
- shaft diameter d_W with tolerance
- bush width B
- viscosity of the fluid at operating temperature.

High temperature operation



Permaglide® P1 swells in some mineral oils if the temperature exceeds +100 °C.

It is sufficient to increase the internal clearance in this case, since other characteristics of Permaglide® P1 are not affected.

Operating behaviour

Running-in process

During the running-in process, part of the running-in layer is transferred to the mating surface, Figure 25 to Figure 27:

- this compensates uneven areas
- a mating surface with a low coefficient of friction is formed, which acts favourably on the operating behaviour
- after running in, some of the porous bronze layer is visible on the sliding layer as individual areas of differing sizes.
This shows:
 - that the bearing is running correctly.

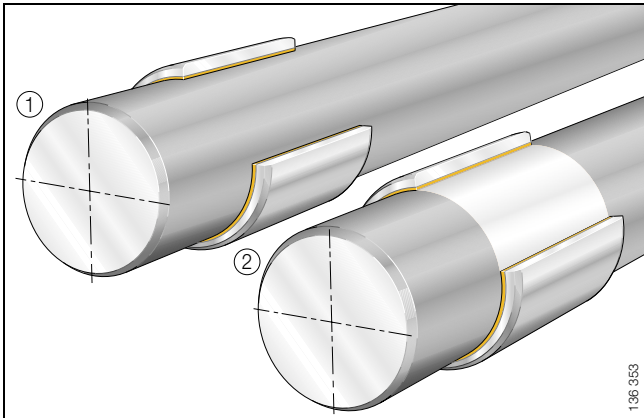
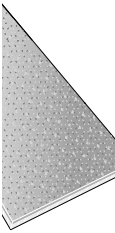


Figure 25 · Material transfer during running-in

- ① Before running-in
- ② After running-in

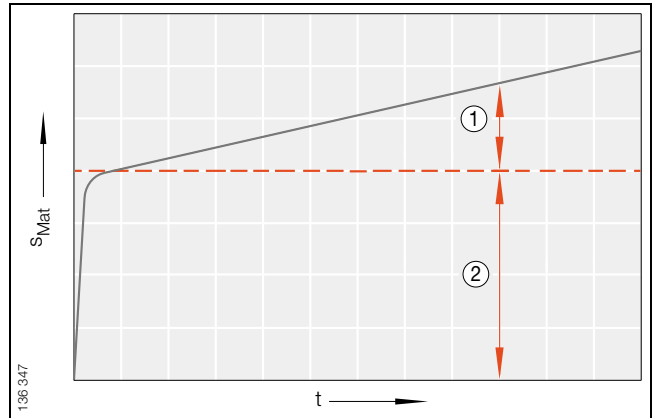


Figure 26 · Typical operating behaviour, material wear s_{Mat} during operating life t

- ① Wear during operation
- ② Material transfer during running-in

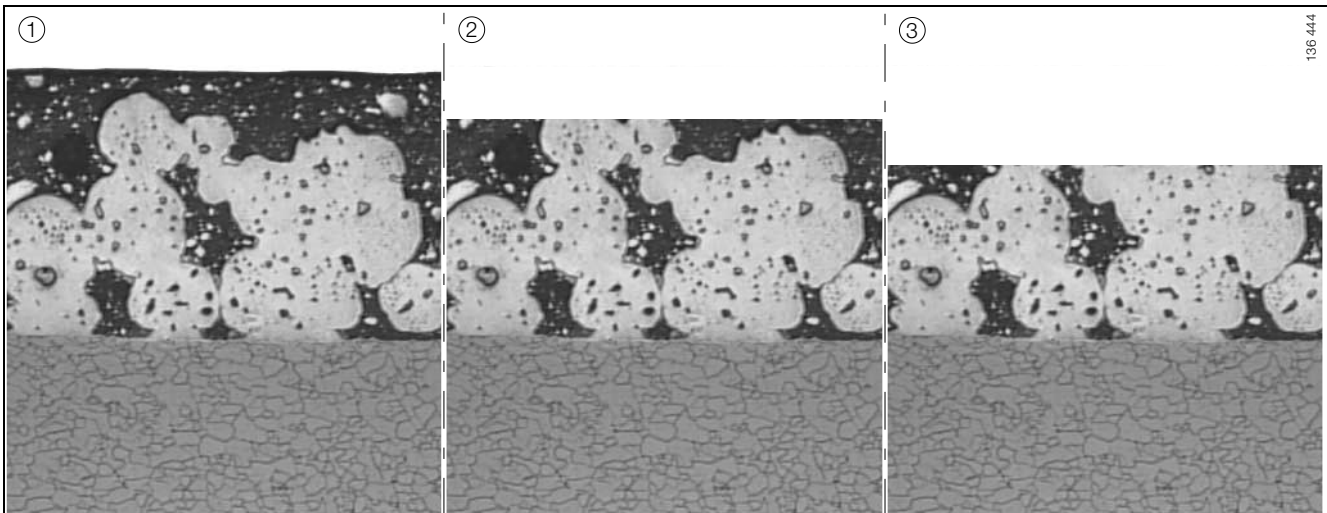


Figure 27 · Permaglide® P10

- ① Before running-in
- ② After running-in
- ③ After long operating life

Sizing

Permaglide® plain bearings are supplied ready-to-fit. Permaglide® bushings should be sized only if there is no other way to achieve a reduced tolerance for the internal clearance.

⚠ Sizing significantly shortens the rating life of Permaglide® bushes P1 (Table 20).

- Figure 28 shows sizing using an arbor
- Table 20 contains guide values for the diameter of the burnishing tool d_K
- Exact values can only be determined by means of tests.

Preferable methods

The internal clearance tolerance can also be reduced using measures that do not shorten the rating life:

- tighter housing bore tolerances
- tighter shaft tolerances.

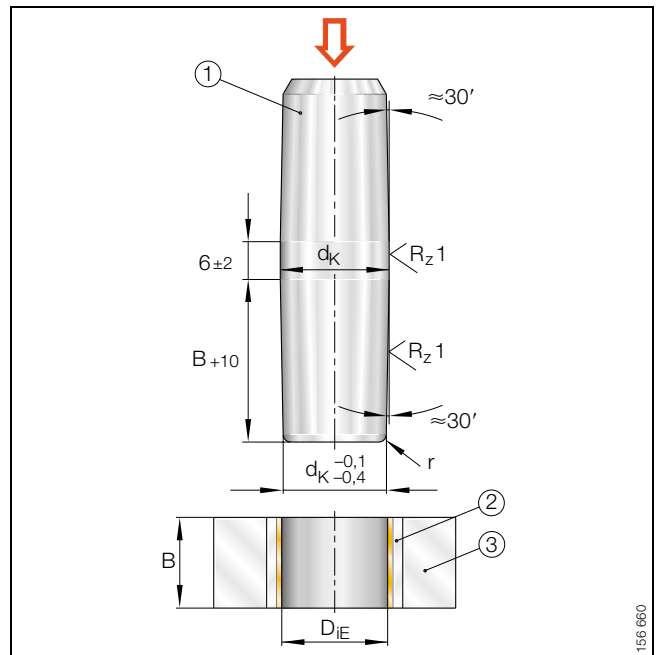


Figure 28 · Sizing

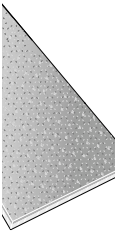
- ① Burnishing tool, case depth $E_{ht} > 0,6$, HRC 56 to 64
- ② Permaglide® bush PAP.P10
- ③ Housing
- B Width of bush
- D_{IE} Diameter of the bush after pressing-in
- d_K Diameter of burnishing tool
- r Rounded edge

Table 20 · Guide values for diameter of burnishing tool and reduction in rating life

Required inside diameter of bush	Diameter of burnishing tool ¹⁾ d_K	Rating life ²⁾
D_{IE}	–	100% L_h
$D_{IE} + 0,02$	$D_{IE} + 0,06$	80% L_h
$D_{IE} + 0,03$	$D_{IE} + 0,08$	60% L_h
$D_{IE} + 0,04$	$D_{IE} + 0,10$	30% L_h

D_{IE} Inside diameter of bush after pressing-in.

- 1) Guide value related to steel housing.
- 2) Guide value for dry running.



Special designs

Special designs

Available by agreement:

- Permaglide® P14
 - lead-free
 - running-in layer and sliding layer resistant to swelling
 - operating temperature range -200 °C to $+280\text{ °C}$
 - identical in design to P10, but with lead-free bronze and zinc sulphide in the running-in layer and sliding layer instead of lead
- Permaglide® P16
 - particularly good wear resistance
 - operating temperature range -40 °C to $+160\text{ °C}$
 - identical in design to P10, but with polyvinylidene fluoride (PVDF) in the running-in layer and sliding layer
- Permaglide® P18
 - particularly good wear resistance
 - preferred for axial movements
 - operating temperature range -200 °C to $+280\text{ °C}$
 - only for use with oil
 - identical in design to P10, but carbon fibres in the running-in layer and sliding layer
- all maintenance-free Permaglide® materials P1 (except for P11) with increased corrosion protection.

Further information

Page



<i>Technical principles</i>	10
<i>Rating life</i>	10
<i>Design of bearing arrangements</i>	19
<i>Internal clearance and mounting tolerances</i>	24
<i>Pressing in of bushes</i>	30
<i>Calculation of press-in force</i>	32

Materials

Permaglide® P2
Low-maintenance plain bearing material



Features 44



Design and safety information 46



Special designs..... 48

Page




Features

Permaglide® plain bearing material P2

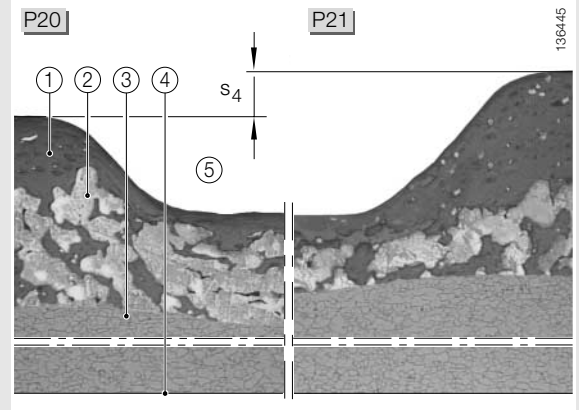
- low maintenance
- suitable for
 - rotating and
 - oscillating movements
- long relubrication intervals
- low wear
- good resistance to edge stress
- good damping behaviour
- good resistance to shock loads.

Variants

- P20
 - with lubrication pockets
 - ready-to-fit
- P21, available by agreement
 - with lubrication pockets
 - with machining allowance.
The sliding layer is on average 0,15 mm thicker than in the case of P20. It can therefore be machined subsequently. Misalignment inaccuracies can therefore be compensated or internal clearances with tighter tolerances can be achieved
- P22, available by agreement
 - without lubrication pockets
 - with machining allowance.
The sliding layer is on average 0,15 mm thicker than in the case of P20. It can therefore be machined subsequently. Misalignment inaccuracies can therefore be compensated or internal clearances with tighter tolerances can be achieved
- P23, available by agreement
 - without lubrication pockets
 - ready-to-fit.

 Permaglide® P20 to P23 contain lead (Pb). Therefore, contact with foodstuffs or pharmaceutical products should be avoided.

Permaglide® plain bearing material, low maintenance

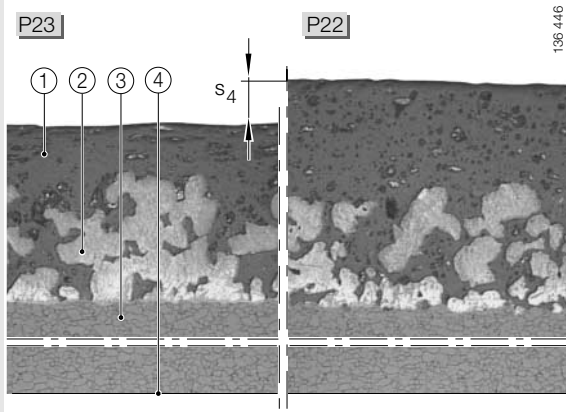


- Sliding layer ①: polyvinylidene fluoride (PVDF), polytetrafluoroethylene (PTFE) and lead (Pb), 0,05 mm to 0,10 mm thick
- Bronze intermediate layer ②: 0,20 mm to 0,35 mm thick
- Steel backing ③
- Surface protection ④: tin, approx. 0,002 mm thick
- P20 and P21 with lubrication pockets ⑤
- P21 with machining allowance s_4 of approx. 0,15 mm

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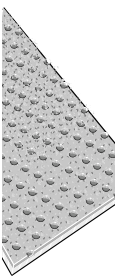
Permaglide® plain bearing material, low maintenance



- Sliding layer ①: polyvinylidene fluoride (PVDF), polytetrafluoroethylene (PTFE) and lead (Pb), 0,05 mm to 0,10 mm thick
- Bronze intermediate layer ②: 0,20 mm to 0,35 mm thick
- Steel backing ③
- Surface protection ④: tin, approx. 0,002 mm thick
- P22 and P23 without lubrication pockets
- P22 with machining allowance s_4 of approx. 0,15 mm

Technical data

Maximum pv value		pv	3	N/mm ² · m/s
Permissible specific bearing load	static	p_{max}	250	N/mm ²
	very low sliding speed	p_{max}	140	N/mm ²
	rotating, oscillating	p_{max}	70	N/mm ²
Permissible sliding speed		v_{max}	3	m/s
	hydrodynamic operation	v_{max}	> 3	m/s
Permissible operating temperature	continuous operation	ϑ	- 40 to +110	°C
	for short periods	ϑ_{max}	+ 140	°C
Coefficient of thermal expansion	steel backing	α_{St}	$11 \cdot 10^{-6}$	K ⁻¹
Coefficient of thermal conductivity	steel backing	λ_{St}	< 4	W (m · K) ⁻¹
Coefficient of friction		μ	0,02 to 0,2	





Design and safety information

Friction

The friction is dependent on:

- the lubricant
- the roughness depth of the mating surface
- the specific bearing load
- the sliding speed
- the operating temperature
- the wear conditions.

Coefficient of friction

$$0,02 < \mu < 0,2$$

Chemical resistance and corrosion protection

The resistance of Permaglide® P2 depends on the chemical characteristics of the individual layers.

In general:

- Permaglide® P2 is resistant to water, alcohols, glycols and many mineral oils.
- With Permaglide® P2, the tin-plated steel surface is adequate protection against corrosion in most cases.
 - special designs, page 48.

Special operating conditions

- Permaglide® P2 is not resistant to acidic (pH <5) and alkaline (pH >9) media.

Fretting corrosion

Rust cannot form between the sliding material and the mating surface due to the sliding layer comprising PVDF, PTFE, Pb and the lubricant.

Fretting corrosion occurs only infrequently between the steel backing of Permaglide® P2 and the housing. Electroplated protective layers delay corrosion in such cases.

Electrochemical contact corrosion

In unfavourable conditions, local elements can form which reduce the operating life.

Remedy:

- note this factor at the design stage
- clarify by means of tests if necessary.

Lubrication

Low-maintenance Permaglide® P2 must be lubricated with grease or fluid.

Permaglide® P20 and P21 have lubrication pockets that retain the lubricant. The initial lubrication is therefore adequate in most cases.

The operating life increases if relubrication is carried out regularly.

The lubricant also protects the bearing arrangement against corrosion.

Lubricating greases

- Lithium soap greases with a mineral oil base are highly suitable
- Grease additives such as molybdenum disulphide and zinc sulphide are unfavourable since they increase wear:
 - greases may contain max. 5% MoS₂.

Hydrodynamic properties

Permaglide® plain bearings P22 and P23 (without lubrication pockets) can be operated under hydrodynamic conditions.

Advantages:

- higher circumferential speeds permissible than with dry running
- wear-free operation:
 - only fluid friction is present once the transition speed is reached.

INA offers the calculation of hydrodynamic operating conditions for Permaglide® bearings as a service.

The following data must be provided:

- load
- speed
- housing bore diameter d_G with tolerance
- shaft diameter d_W with tolerance
- bush width B
- viscosity of the fluid at operating temperature.

Machining of the sliding layer

The sliding layers of Permaglide® P21 and P22 have a machining allowance of approx. 0,15 mm suitable for

- turning, drilling or reaming, in order to
 - achieve narrower clearance tolerances
 - compensate for misalignment inaccuracies.
- the following have proven effective for turning and drilling:
 - dry cutting
 - cutting speed 100 m/min to 150 m/min
 - feed 0,05 mm
 - maximum cutting depth 0,1 mm
 - hard metal cutting tools (Figure 29).



Removing a substantial amount of material reduces the operating life and the lubrication pocket volume.

Incorrect machining will have a detrimental effect on the operating life and the load carrying capacity.

Parts must be cleaned after machining.

During machining, burrs can form in the area of the lubrication pockets (Permaglide® P21).

Exceeding a temperature of +140 °C during machining constitutes a health hazard.

The swarf produced may contain lead.

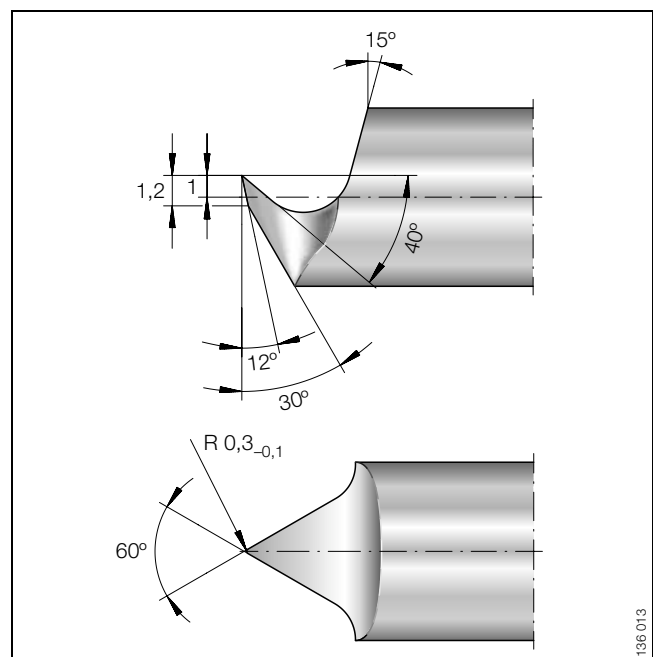
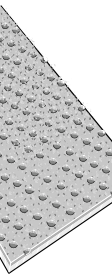


Figure 29 · Cutting tool for Permaglide® P21 and P22




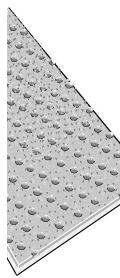
Special designs

Special designs

Available by agreement:

- Permaglide® P25
 - with lubrication pockets
 - ready-to-fit
 - with bronze backing, therefore substantially corrosion-resistant
- all low-maintenance Permaglide® materials P2 (except for P25) with increased corrosion protection.

Further information	Page
 <i>Technical principles</i>	10
<i>Rating life</i>	10
<i>Design of bearing arrangements</i>	19
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Designs

Maintenance-free Permaglide® plain bearings P1
 Low-maintenance Permaglide® plain bearings P2



Features 50



Ordering example and ordering designation 52



Dimension table 53

Page



Features

Maintenance-free Permaglide® plain bearings P1

- Suitable primarily for dry running
- Technical data, page 37
 - $pV_{max} = 1,8 \text{ N/mm}^2 \cdot \text{m/s}$
 - $pV_{temp} = 3,6 \text{ N/mm}^2 \cdot \text{m/s}$
 - $p_{max \text{ stat.}} = 250 \text{ N/mm}^2$
 - $p_{max \text{ dyn.}} = 56 \text{ N/mm}^2$
 - $v_{max} = 2 \text{ m/s}$
 - $\vartheta = -200 \text{ }^\circ\text{C to } +280 \text{ }^\circ\text{C}$
- Permaglide® P10 with steel backing
- Permaglide® P10 with bronze backing.

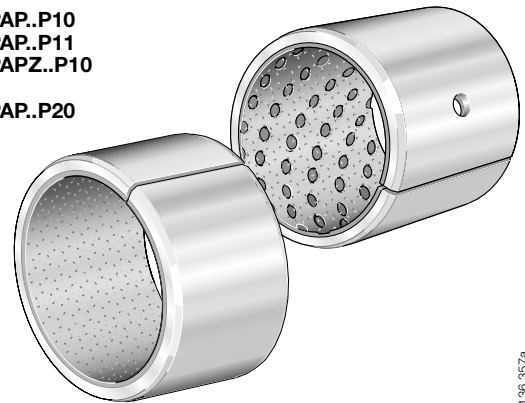
Low-maintenance Permaglide® plain bearings P2

- Lubrication required
- Technical data, page 45
 - $pV_{max} = 3 \text{ N/mm}^2 \cdot \text{m/s}$
 - $p_{max \text{ stat.}} = 250 \text{ N/mm}^2$
 - $p_{max \text{ dyn.}} = 70 \text{ N/mm}^2$
 - $v_{max} = -3 \text{ m/s}$
 - $\vartheta = -40 \text{ }^\circ\text{C to } +110 \text{ }^\circ\text{C}$
 - $\vartheta_{max} = \text{up to } +140 \text{ }^\circ\text{C for short periods}$
- Permaglide® P20 with lubrication pockets.

Bushes PAP

PAP..P10
 PAP..P11
 PAPZ..P10

PAP..P20

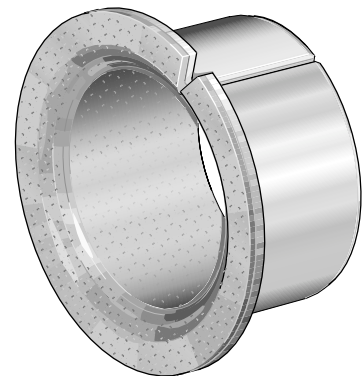


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- PAP..P10 for shafts from 2 mm to 300 mm
- PAP..P11 for shafts from 4 mm to 100 mm
- PAPZ..P10 for shafts from 3/16" to 2", inch sizes
- PAP..P20 for shafts from 8 mm to 100 mm

Flanged bushes PAF

PAF..P10
 PAF..P11



136 360

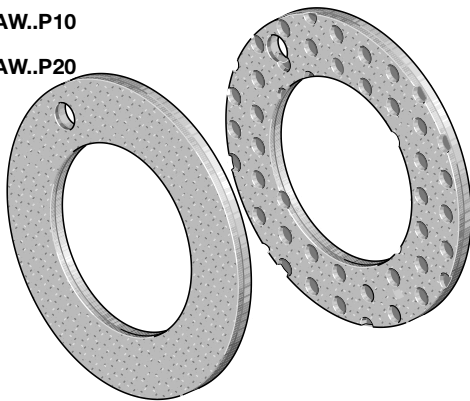
- PAF..P10 for shafts from 6 mm to 40 mm
- PAF..P11 for shafts from 6 mm to 40 mm



Thrust washers PAW

PAW..P10

PAW..P20



136 361

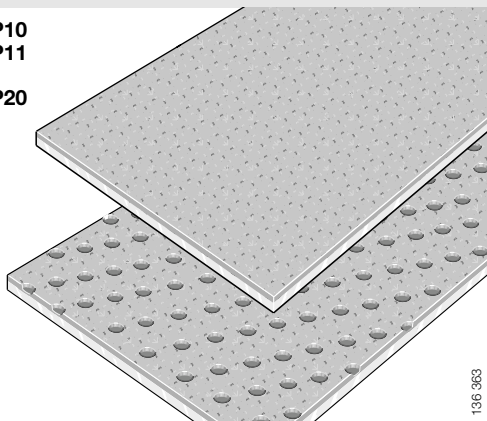
- PAW..P10 with inside diameter from 10 mm to 62 mm
- PAW..P11 available by agreement
- PAW..P20 with inside diameter from 12 mm to 52 mm

Strips PAS

PAS..P10

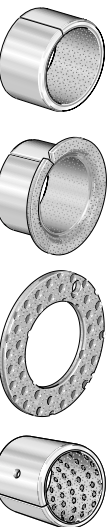
PAS..P11

PAS..P20



136 363

- PAS..P10, PAS..P11
 - Length 500 mm
 - Widths, see *dimension tables* page 62
 - Wall thicknesses, see *dimension tables* page 62
- PAS..P20
 - Length 500 mm
 - Width 180 mm
 - Wall thicknesses, see *dimension tables* page 65



Designs

Ordering examples



Ordering example and ordering designation

Bush, Permagliding® P10
with steel backing:

Inside diameter 16 mm
Width 25 mm

Ordering designation: PAP 1625 P10

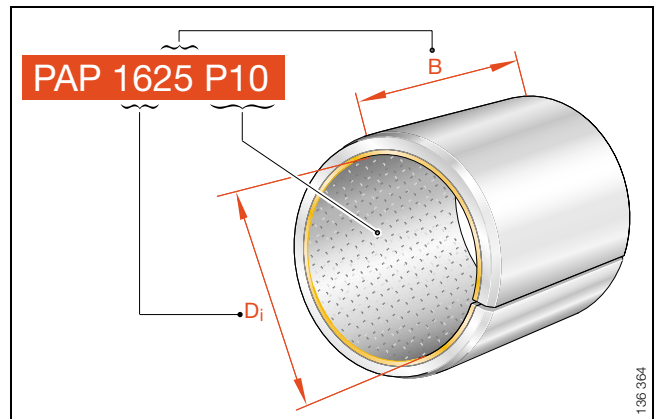


Figure 30 · Ordering example, bush P10

Strip, Permagliding® P20:

Length 500 mm
Width 180 mm
Wall thickness 1 mm (ordering data: $s_3 \times 10$)

Ordering designation: PAS 10180 P20

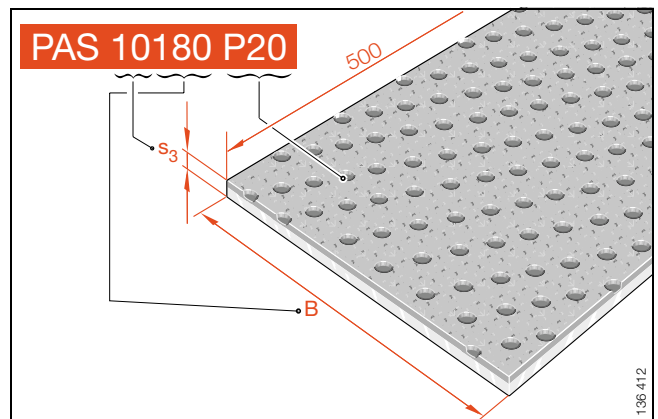


Figure 31 · Ordering example, strip P20

Thrust washers, Permagliding® P20:

Inside diameter 12 mm

Ordering designation: PAW 12 P20

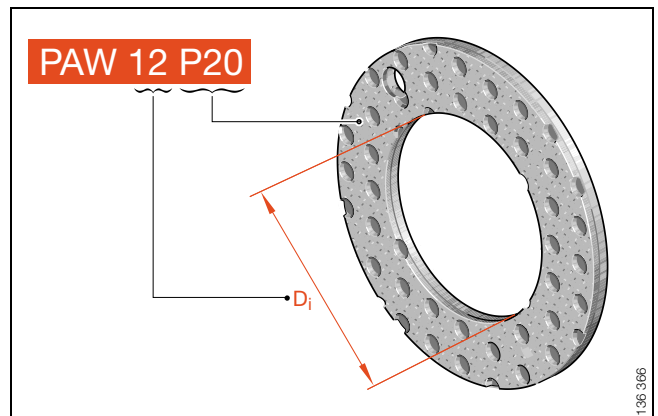


Figure 32 · Ordering example, thrust washer P20

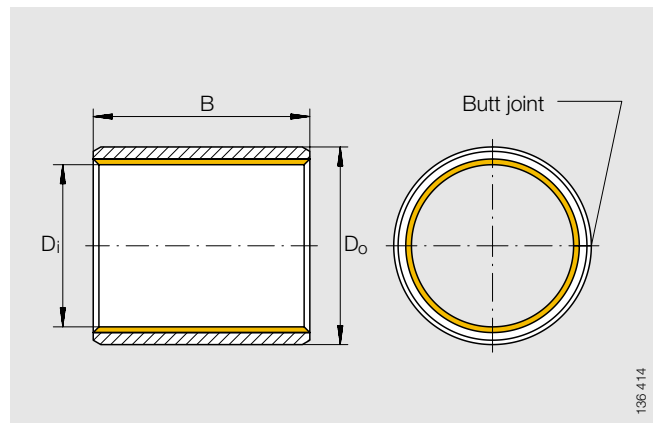
Dimension tables

Permaglide®

Bushes

Maintenance-free, with steel backing

Series PAP..P10



PAP

Dimension table · Dimensions in mm					
Shaft diameter	Designation	Mass g	Dimensions		
			D _i	D _o	B ±0,25
2	PAP 0203 P10	0,15	2	3,5	3
	PAP 0205 P10	0,25	2	3,5	5
3	PAP 0303 P10	0,2	3	4,5	3
	PAP 0304 P10	0,25	3	4,5	4
	PAP 0305 P10	0,3	3	4,5	5
	PAP 0306 P10	0,4	3	4,5	6
4	PAP 0403 P10	0,2	4	5,5	3
	PAP 0404 P10	0,3	4	5,5	4
	PAP 0406 P10	0,6	4	5,5	6
	PAP 0410 P10	0,8	4	5,5	10
5	PAP 0505 P10	0,7	5	7	5
	PAP 0508 P10	1,1	5	7	8
	PAP 0510 P10	1,4	5	7	10
6	PAP 0606 P10	0,9	6	8	6
	PAP 0608 P10	1,4	6	8	8
	PAP 0610 P10	1,7	6	8	10
7	PAP 0710 P10	1,8	7	9	10
8	PAP 0808 P10	1,7	8	10	8
	PAP 0810 P10	2,1	8	10	10
	PAP 0812 P10	2,5	8	10	12
10	PAP 1008 P10	2	10	12	8
	PAP 1010 P10	2,5	10	12	10
	PAP 1012 P10	2,9	10	12	12
	PAP 1015 P10	3,8	10	12	15
	PAP 1020 P10	5,3	10	12	20

Recommended mounting tolerance:

Shaft	Housing
$d_w < 5$: h6	$d_G \leq 5,5$: H6
$5 \leq d_w < 80$: f7	$5,5 < d_G$: H7
$80 \leq d_w$: h8	

Internal clearances, wall thicknesses and chamfer tolerances:
see page 25ff.

Bushes in special sizes are available by agreement.

Dimension table (continued) · Dimensions in mm					
Shaft diameter	Designation	Mass g	Dimensions		
			D _i	D _o	B ±0,25
12	PAP 1208 P10	2	12	14	8
	PAP 1210 P10	3	12	14	10
	PAP 1212 P10	3,7	12	14	12
	PAP 1215 P10	4,7	12	14	15
	PAP 1220 P10	6,1	12	14	20
	PAP 1225 P10	7,6	12	14	25
13	PAP 1310 P10	3,2	13	15	10
14	PAP 1410 P10	3,5	14	16	10
	PAP 1412 P10	4,3	14	16	12
	PAP 1415 P10	5,4	14	16	15
	PAP 1420 P10	7,1	14	16	20
	PAP 1425 P10	8,8	14	16	25
15	PAP 1510 P10	3,7	15	17	10
	PAP 1512 P10	4,5	15	17	12
	PAP 1515 P10	5,7	15	17	15
	PAP 1520 P10	7,6	15	17	20
	PAP 1525 P10	9,4	15	17	25
16	PAP 1610 P10	4	16	18	10
	PAP 1612 P10	4,8	16	18	12
	PAP 1615 P10	6,1	16	18	15
	PAP 1620 P10	8,1	16	18	20
	PAP 1625 P10	10,1	16	18	25
18	PAP 1810 P10	4,5	18	20	10
	PAP 1815 P10	6,7	18	20	15
	PAP 1820 P10	8,9	18	20	20
	PAP 1825 P10	11,1	18	20	25



Bushes

Maintenance-free, with steel backing

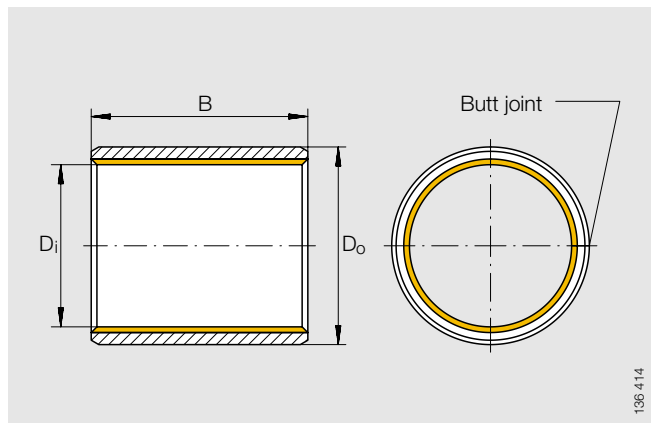
Series PAP..P10

Dimension table (continued) · Dimensions in mm					
Shaft diameter	Designation	Mass g	Dimensions		
			D _i	D _o	B ±0,25
20	PAP 2010 P10	7,7	20	23	10
	PAP 2015 P10	11,1	20	23	15
	PAP 2020 P10	15,1	20	23	20
	PAP 2025 P10	19,1	20	23	25
	PAP 2030 P10	23	20	23	30
22	PAP 2215 P10	12,7	22	25	15
	PAP 2220 P10	16,6	22	25	20
	PAP 2225 P10	21,1	22	25	25
	PAP 2230 P10	25,2	22	25	30
24	PAP 2415 P10	13,5	24	27	15
	PAP 2420 P10	17,9	24	27	20
	PAP 2425 P10	22,8	24	27	25
	PAP 2430 P10	27,1	24	27	30
25	PAP 2510 P10	9,6	25	28	10
	PAP 2515 P10	14,2	25	28	15
	PAP 2520 P10	19	25	28	20
	PAP 2525 P10	23,9	25	28	25
	PAP 2530 P10	28,4	25	28	30
	PAP 2540 P10	38	25	28	40
	PAP 2550 P10	47,7	25	28	50
28	PAP 2820 P10	28,8	28	32	20
	PAP 2830 P10	44	28	32	30

Recommended mounting tolerance:
 Shaft $d_W < 5$: h6 Housing $d_G \leq 5,5$: H6
 $5 \leq d_W < 80$: f7 $5,5 < d_G$: H7
 $80 \leq d_W$: h8 h8

Internal clearances, wall thicknesses and chamfer tolerances:
 see page 25 ff.
 Bushes in special sizes are available by agreement.

Dimension table (continued) · Dimensions in mm					
Shaft diameter	Designation	Mass g	Dimensions		
			D _i	D _o	B ±0,25
30	PAP 3015 P10	22,9	30	34	15
	PAP 3020 P10	30,9	30	34	20
	PAP 3025 P10	38,5	30	34	25
	PAP 3030 P10	46,1	30	34	30
	PAP 3040 P10	63	30	34	40
	PAP 3050 P10	81	30	34	50
32	PAP 3230 P10	49	32	36	30
	PAP 3240 P10	65	32	36	40
35	PAP 3520 P10	35	35	39	20
	PAP 3530 P10	53	35	39	30
	PAP 3540 P10	71	35	39	40
40	PAP 4020 P10	40	40	44	20
	PAP 4030 P10	60	40	44	30
	PAP 4040 P10	81	40	44	40
45	PAP 4530 P10	86	45	50	30
	PAP 4540 P10	113	45	50	40
	PAP 4550 P10	144	45	50	50
50	PAP 5020 P10	63	50	55	20
	PAP 5030 P10	95	50	55	30
	PAP 5040 P10	127	50	55	40
	PAP 5060 P10	188	50	55	60
55	PAP 5540 P10	138	55	60	40
	PAP 5560 P10	207	55	60	60
60	PAP 6030 P10	113	60	65	30
	PAP 6040 P10	150	60	65	40
	PAP 6060 P10	226	60	65	60
	PAP 6070 P10	265	60	65	70
65	PAP 6530 P10	123	65	70	30
	PAP 6540 P10	164	65	70	40
	PAP 6550 P10	204	65	70	50
	PAP 6560 P10	244	65	70	60
	PAP 6570 P10	284	65	70	70



PAP

Dimension table (continued) · Dimensions in mm

Shaft diameter	Designation	Mass g	Dimensions		
			D_i	D_o	B $\pm 0,25$
70	PAP 7040 P10	174	70	75	40
	PAP 7050 P10	218	70	75	50
	PAP 7070 P10	305	70	75	70
75	PAP 7540 P10	187	75	80	40
	PAP 7550 P10	233	75	80	50
	PAP 7560 P10	280	75	80	60
	PAP 7580 P10	374	75	80	80
80	PAP 8040 P10	197	80	85	40
	PAP 8060 P10	297	80	85	60
	PAP 8080 P10	395	80	85	80
	PAP 80100 P10	493	80	85	100
85	PAP 8560 P10	313	85	90	60
	PAP 85100 P10	525	85	90	100
90	PAP 9050 P10	277	90	95	50
	PAP 9060 P10	333	90	95	60
	PAP 90100 P10	551	90	95	100
95	PAP 9560 P10	351	95	100	60
	PAP 95100 P10	583	95	100	100
100	PAP 10050 P10	312	100	105	50
	PAP 10060 P10	388	100	105	60
	PAP 100115 P10	742	100	105	115
105	PAP 10560 P10	370	105	110	60
	PAP 105115 P10	712	105	110	115
110	PAP 11060 P10	410	110	115	60
	PAP 110115 P10	775	110	115	115
115	PAP 11550 P10	350	115	120	50
	PAP 11560 P10	400	115	120	60
	PAP 11570 P10	450	115	120	70

Recommended mounting tolerance:

Shaft	Housing
$d_w < 5$: h6	$d_G \leq 5,5$: H6
$5 \leq d_w < 80$: f7	$5,5 < d_G$: H7
$80 \leq d_w$: h8	

Internal clearances, wall thicknesses and chamfer tolerances:
see page 25ff.

Bushes in special sizes are available by agreement.

Dimension table (continued) · Dimensions in mm

Shaft diameter	Designation	Mass g	Dimensions		
			D_i	D_o	B $\pm 0,25$
120	PAP 12060 P10	435	120	125	60
	PAP 120100 P10	730	120	125	100
125	PAP 125100 P10	760	125	130	100
130	PAP 13060 P10	470	130	135	60
	PAP 130100 P10	795	130	135	100
135	PAP 13560 P10	490	135	140	60
	PAP 13580 P10	652	135	140	80
140	PAP 14060 P10	515	140	145	60
	PAP 140100 P10	855	140	145	100
150	PAP 15060 P10	550	150	155	60
	PAP 15080 P10	730	150	155	80
	PAP 150100 P10	915	150	155	100
160	PAP 16080 P10	776	160	165	80
	PAP 160100 P10	970	160	165	100
180	PAP 180100 P10	1100	180	185	100
200	PAP 200100 P10	1220	200	205	100
220	PAP 220100 P10	1320	220	225	100
250	PAP 250100 P10	1495	250	255	100
300	PAP 300100 P10	1760	300	305	100



Bushes

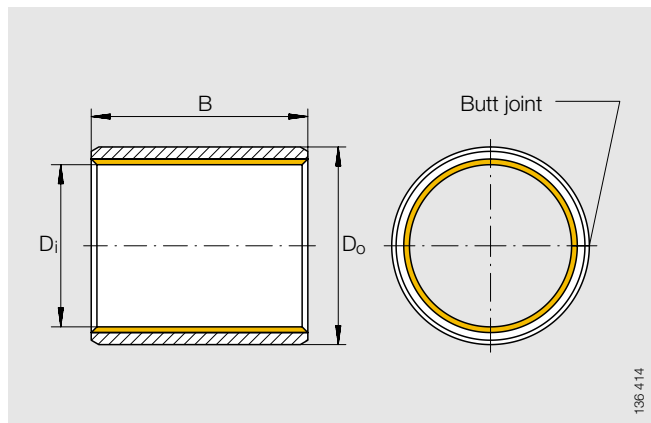
Maintenance-free, with steel backing
Inch sizes

Series PAPZ..P10

Dimension table · Dimensions in <i>inch/mm</i>						
Shaft diameter	Designation	Mass g	Dimensions			
			D _i	D _o	B	
³/₁₆ 4,763	PAPZ 0303 P10	0,5	³ / ₁₆	¹ / ₄	³ / ₁₆	
			4,763	6,35	4,76±0,25	
	PAPZ 0304 P10	0,7	³ / ₁₆	¹ / ₄	¹ / ₄	
			4,763	6,35	6,35±0,25	
	PAPZ 0306 P10	1	³ / ₁₆	¹ / ₄	³ / ₈	
			4,763	6,35	9,53±0,25	
¹/₄ 6,35	PAPZ 0404 P10	0,8	¹ / ₄	⁵ / ₁₆	¹ / ₄	
			6,35	7,938	6,35±0,25	
	PAPZ 0406 P10	1,3	¹ / ₄	⁵ / ₁₆	³ / ₈	
			6,35	7,938	9,53±0,25	
	PAPZ 0408 P10	1,7	¹ / ₄	⁵ / ₁₆	¹ / ₂	
			6,35	7,938	12,7±0,25	
⁵/₁₆ 7,938	PAPZ 0504 P10	1	⁵ / ₁₆	³ / ₈	¹ / ₄	
			7,938	9,525	6,35±0,25	
	PAPZ 0506 P10	1,6	⁵ / ₁₆	³ / ₈	³ / ₈	
			7,938	9,525	9,53±0,25	
	³/₈ 9,525	PAPZ 0603 P10	1,4	³ / ₈	¹⁵ / ₃₂	³ / ₁₆
				9,525	11,906	4,76±0,25
PAPZ 0604 P10		1,9	³ / ₈	¹⁵ / ₃₂	¹ / ₄	
			9,525	11,906	6,35±0,25	
PAPZ 0606 P10		2,9	³ / ₈	¹⁵ / ₃₂	³ / ₈	
			9,525	11,906	9,53±0,25	
PAPZ 0608 P10	3,8	³ / ₈	¹⁵ / ₃₂	¹ / ₂		
		9,525	11,906	12,70±0,25		
PAPZ 0610 P10	4,8	³ / ₈	¹⁵ / ₃₂	⁵ / ₈		
		9,525	11,906	15,88±0,25		
PAPZ 0612 P10	5,8	³ / ₈	¹⁵ / ₃₂	³ / ₄		
		9,525	11,906	19,05±0,25		

Recommended mounting tolerances, wall thicknesses, internal clearances and chamfer tolerances: see page 28ff.
Bushes in special sizes are available by agreement.

Dimension table (continued) · Dimensions in <i>inch/mm</i>					
Shaft diameter	Designation	Mass g	Dimensions		
			D _i	D _o	B
⁷/₁₆ 11,113	PAPZ 0706 P10	3,3	⁷ / ₁₆	¹⁷ / ₃₂	³ / ₈
			11,113	13,494	9,53±0,25
	PAPZ 0708 P10	4,4	⁷ / ₁₆	¹⁷ / ₃₂	¹ / ₂
			11,113	13,494	12,70±0,25
	PAPZ 0710 P10	5,5	⁷ / ₁₆	¹⁷ / ₃₂	⁵ / ₈
			11,113	13,494	15,88±0,25
PAPZ 0712 P10	6,6	⁷ / ₁₆	¹⁷ / ₃₂	³ / ₄	
		11,113	13,494	19,05±0,25	
¹/₂ 12,7	PAPZ 0804 P10	2,5	¹ / ₂	¹⁹ / ₃₂	¹ / ₄
			12,7	15,081	6,35±0,25
	PAPZ 0806 P10	3,7	¹ / ₂	¹⁹ / ₃₂	³ / ₈
			12,7	15,081	9,53±0,25
	PAPZ 0808 P10	5	¹ / ₂	¹⁹ / ₃₂	¹ / ₂
			12,7	15,081	12,70±0,25
PAPZ 0810 P10	6,2	¹ / ₂	¹⁹ / ₃₂	⁵ / ₈	
		12,7	15,081	15,88±0,25	
PAPZ 0812 P10	7,5	¹ / ₂	¹⁹ / ₃₂	³ / ₄	
		12,7	15,081	19,05±0,25	
PAPZ 0814 P10	8,7	¹ / ₂	¹⁹ / ₃₂	⁷ / ₈	
		12,7	15,081	22,23±0,25	
⁹/₁₆ 14,288	PAPZ 0906 P10	4	⁹ / ₁₆	²¹ / ₃₂	³ / ₈
			14,288	16,669	9,53±0,25
	PAPZ 0908 P10	5,5	⁹ / ₁₆	²¹ / ₃₂	¹ / ₂
			14,288	16,669	12,70±0,25
	PAPZ 0912 P10	8,3	⁹ / ₁₆	²¹ / ₃₂	³ / ₄
			14,288	16,669	19,05±0,25
⁵/₈ 15,875	PAPZ 1004 P10	3	⁵ / ₈	²³ / ₃₂	¹ / ₄
			15,875	18,256	6,35±0,25
	PAPZ 1008 P10	6,1	⁵ / ₈	²³ / ₃₂	¹ / ₂
			15,875	18,256	12,70±0,25
	PAPZ 1010 P10	7,6	⁵ / ₈	²³ / ₃₂	⁵ / ₈
			15,875	18,256	15,88±0,25
PAPZ 1012 P10	9,2	⁵ / ₈	²³ / ₃₂	³ / ₄	
		15,875	18,256	19,05±0,25	
PAPZ 1014 P10	10,7	⁵ / ₈	²³ / ₃₂	⁷ / ₈	
		15,875	18,256	22,23±0,25	



PAPZ

Dimension table (continued) · Dimensions in inch/mm

Shaft diameter	Designation	Mass g	Dimensions		
			Di	Do	B
11/16 17,463	PAPZ 1112 P10	10	11/16	25/32	3/4
			17,463	19,844	19,05±0,25
3/4 19,05	PAPZ 1204 P10	4,9	3/4	7/8	1/4
			19,05	22,225	6,35±0,25
	PAPZ 1206 P10	7,4	3/4	7/8	3/8
			19,05	22,225	9,53±0,25
	PAPZ 1208 P10	9,9	3/4	7/8	1/2
			19,05	22,225	12,70±0,25
	PAPZ 1210 P10	12,3	3/4	7/8	5/8
			19,05	22,225	15,88±0,25
PAPZ 1212 P10	14,8	3/4	7/8	3/4	
		19,05	22,225	19,05±0,25	
PAPZ 1216 P10	19,7	3/4	7/8	1	
		19,05	22,225	25,40±0,25	
7/8 22,225	PAPZ 1412 P10	17,1	7/8	1	3/4
			22,225	25,4	19,05±0,25
PAPZ 1416 P10	22,8	7/8	1	1	
		22,225	25,4	25,40±0,25	
1 25,4	PAPZ 1606 P10	9,7	1	1 1/8	3/8
			25,4	28,575	9,53±0,25
	PAPZ 1608 P10	12,9	1	1 1/8	1/2
			25,4	28,575	12,70±0,25
	PAPZ 1612 P10	19,3	1	1 1/8	3/4
			25,4	28,575	19,05±0,25
	PAPZ 1614 P10	22,6	1	1 1/8	7/8
			25,4	28,575	22,23±0,25
PAPZ 1616 P10	25,8	1	1 1/8	1	
		25,4	28,575	25,40±0,25	
PAPZ 1620 P10	32,2	1	1 1/8	1 1/4	
		25,4	28,575	31,75±0,25	
PAPZ 1624 P10	38,7	1	1 1/8	1 1/2	
		25,4	28,575	38,10±0,25	

Recommended mounting tolerances, wall thicknesses, internal clearances and chamfer tolerances: see page 28ff.
Bushes in special sizes are available by agreement.

Dimension table (continued) · Dimensions in inch/mm

Shaft diameter	Designation	Mass g	Dimensions		
			Di	Do	B
1 1/8 28,575	PAPZ 1808 P10	18,3	1 1/8	1 9/32	1/2
			28,575	32,544	12,70±0,25
	PAPZ 1812 P10	27,4	1 1/8	1 9/32	3/4
PAPZ 1816 P10	36,5	1 1/8	1 9/32	1	
		28,575	32,544	25,40±0,25	
1 1/4 31,75	PAPZ 2006 P10	15,1	1 1/4	1 13/32	3/8
			31,75	35,719	9,53±0,25
	PAPZ 2012 P10	30,2	1 1/4	1 13/32	3/4
			31,75	35,719	19,05±0,25
	PAPZ 2016 P10	40,3	1 1/4	1 13/32	1
			31,75	35,719	25,40±0,25
PAPZ 2020 P10	50,4	1 1/4	1 13/32	1 1/4	
		31,75	35,719	31,75±0,25	
1 3/8 34,925	PAPZ 2206 P10	16,5	1 3/8	1 17/32	3/8
			34,925	38,894	9,53±0,25
	PAPZ 2208 P10	22,1	1 3/8	1 17/32	1/2
			34,925	38,894	12,70±0,25
	PAPZ 2210 P10	27,6	1 3/8	1 17/32	5/8
			34,925	38,894	15,88±0,25
	PAPZ 2212 P10	33	1 3/8	1 17/32	3/4
			34,925	38,894	19,05±0,25
PAPZ 2216 P10	44,1	1 3/8	1 17/32	1	
		34,925	38,894	25,40±0,25	
PAPZ 2224 P10	66	1 3/8	1 17/32	1 1/2	
		34,925	38,894	38,10±0,25	
PAPZ 2228 P10	77	1 3/8	1 17/32	1 3/4	
		34,925	38,894	44,45±0,25	

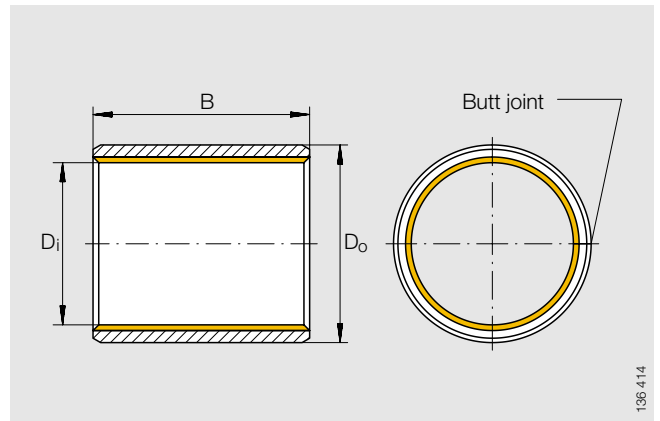


Permaglide®

Bushes

Maintenance-free, with steel backing
Inch sizes

Series PAPZ..P10



PAPZ

Dimension table (continued) · Dimensions in <i>inch/mm</i>					
Shaft diameter	Designation	Mass g	Dimensions		
			Di	Do	B
1 1/2 38,1	PAPZ 2408 P10	23,9	1 1/2	1 21/32	1/2
			38,1	42,069	12,70±0,25
	PAPZ 2416 P10	48	1 1/2	1 21/32	1
			38,1	42,069	25,40±0,25
	PAPZ 2420 P10	60	1 1/2	1 21/32	1 1/4
			38,1	42,069	31,75±0,25
	PAPZ 2424 P10	72	1 1/2	1 21/32	1 1/2
			38,1	42,069	38,10±0,25
	PAPZ 2432 P10	96	1 1/2	1 21/32	2
			38,1	42,069	50,80±0,25
1 5/8 41,275	PAPZ 2616 P10	52	1 5/8	1 25/32	1
			41,275	45,244	25,40±0,25
	PAPZ 2624 P10	78	1 5/8	1 25/32	1 1/2
			41,275	45,244	38,10±0,25
1 3/4 44,45	PAPZ 2816 P10	67	1 3/4	1 15/16	1
			44,45	49,213	25,40±0,25
	PAPZ 2824 P10	101	1 3/4	1 15/16	1 1/2
			44,45	49,213	38,10±0,25
	PAPZ 2832 P10	134	1 3/4	1 15/16	2
			44,45	49,213	50,80±0,25

Recommended mounting tolerances, wall thicknesses, internal clearances and chamfer tolerances: see page 28ff.

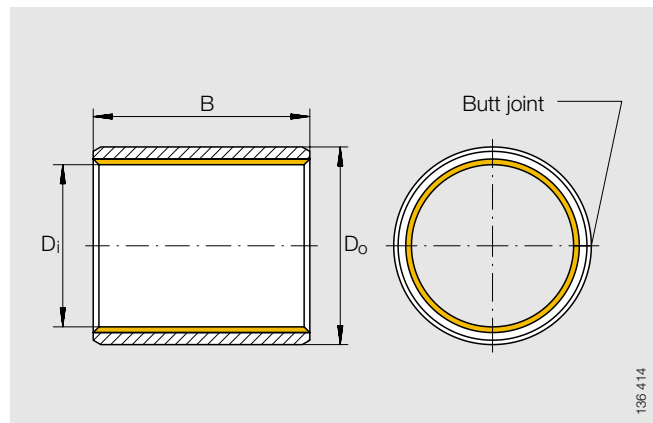
Bushes in special sizes are available by agreement.

Dimension table (continued) · Dimensions in <i>inch/mm</i>					
Shaft diameter	Designation	Mass g	Dimensions		
			Di	Do	B
2 50,8	PAPZ 3216 P10	76	2	2 3/16	1
			50,8	55,563	25,4±0,25
	PAPZ 3224 P10	114	2	2 3/16	1 1/2
			50,8	55,563	38,1±0,25
	PAPZ 3232 P10	153	2	2 3/16	2
			50,8	55,563	50,8±0,25
	PAPZ 3240 P10	191	2	2 3/16	2 1/2
			50,8	55,563	63,5±0,25

Bushes

Maintenance-free, with bronze backing

Series PAP..P11



PAP

Dimension table · Dimensions in mm					
Shaft diameter	Designation	Mass g	Dimensions		
			Di	Do	B ±0,25
4	PAP 0406 P11	0,8	4	6	6
5	PAP 0505 P11	0,8	5	7	5
6	PAP 0606 P11	1,1	6	8	6
	PAP 0610 P11	1,9	6	8	10
8	PAP 0808 P11	1,8	8	10	8
	PAP 0810 P11	2,2	8	10	10
	PAP 0812 P11	2,7	8	10	12
10	PAP 1005 P11	1,4	10	12	5
	PAP 1010 P11	2,7	10	12	10
	PAP 1015 P11	4,2	10	12	15
	PAP 1020 P11	5,6	10	12	20
12	PAP 1210 P11	3,3	12	14	10
	PAP 1212 P11	4,1	12	14	12
	PAP 1215 P11	5,2	12	14	15
	PAP 1220 P11	6,9	12	14	20
	PAP 1225 P11	8,7	12	14	25
14	PAP 1415 P11	6	14	16	15
15	PAP 1515 P11	6,8	15	17	15
	PAP 1525 P11	10,3	15	17	25
16	PAP 1615 P11	6,7	16	18	15
	PAP 1625 P11	11	16	18	25
18	PAP 1815 P11	7,4	18	20	15
	PAP 1825 P11	12,2	18	20	25

Recommended mounting tolerance:

Shaft Housing bore
 $d_w < 80$: f7 H7
 $d_w \geq 80$: h8

Internal clearances, wall thicknesses and chamfer tolerances:
 see page 25ff.

Bushes in special sizes are available by agreement.

Dimension table (continued) · Dimensions in mm					
Shaft diameter	Designation	Mass g	Dimensions		
			Di	Do	B ±0,25
20	PAP 2015 P11	13,3	20	23	15
	PAP 2020 P11	17	20	23	20
	PAP 2025 P11	21,3	20	23	25
	PAP 2030 P11	25,5	20	23	30
22	PAP 2215 P11	15,7	22	25	15
	PAP 2220 P11	20,9	22	25	20
	PAP 2225 P11	26,5	22	25	25
24	PAP 2430 P11	34,1	24	27	30
25	PAP 2525 P11	29,4	25	28	25
	PAP 2530 P11	35,3	25	28	30
28	PAP 2830 P11	46	28	32	30
30	PAP 3020 P11	36,8	30	34	20
	PAP 3030 P11	55	30	34	30
	PAP 3040 P11	74	30	34	40
35	PAP 3520 P11	44	35	39	20
	PAP 3530 P11	66	35	39	30
40	PAP 4050 P11	119	40	44	50
45	PAP 4550 P11	186	45	50	50
50	PAP 5030 P11	134	50	55	30
	PAP 5040 P11	179	50	55	40
	PAP 5060 P11	269	50	55	60
55	PAP 5540 P11	155	55	60	40
60	PAP 6040 P11	168	60	65	40
	PAP 6050 P11	211	60	65	50
	PAP 6060 P11	253	60	65	60
	PAP 6070 P11	295	60	65	70
70	PAP 7050 P11	245	70	75	50
	PAP 7070 P11	342	70	75	70
80	PAP 8060 P11	332	80	85	60
	PAP 80100 P11	554	80	85	100
90	PAP 9060 P11	353	90	95	60
	PAP 90100 P11	588	90	95	100
95	PAP 9560 P11	430	95	100	60
100	PAP 10060 P11	412	100	105	60
	PAP 100115 P11	790	100	105	115



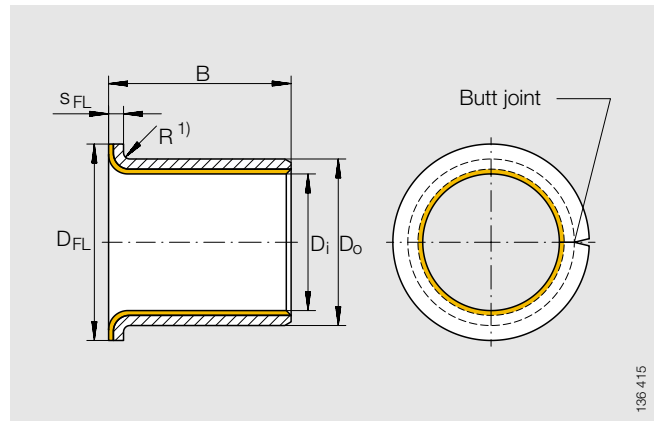
Flanged bushes

Maintenance-free, with steel backing

Series PAF..P10

Maintenance-free, with bronze backing

Series PAF..P11



PAF

Dimension table · Dimensions in mm							
Shaft diameter	Designation PAF..P10	Mass g	Dimensions				
			D _i	D _o	D _{FL} ±0,5	B ±0,25	S _{FL} -0,2
6	PAF 06040 P10	0,9	6	8	12	4	1
	PAF 06070 P10	1,6	6	8	12	7	1
	PAF 06080 P10	1,7	6	8	12	8	1
8	PAF 08055 P10	1,7	8	10	15	5,5	1
	PAF 08075 P10	2,1	8	10	15	7,5	1
	PAF 08095 P10	2,5	8	10	15	9,5	1
10	PAF 10070 P10	2,5	10	12	18	7	1
	PAF 10090 P10	2,9	10	12	18	9	1
	PAF 10120 P10	3,8	10	12	18	12	1
	PAF 10170 P10	5,4	10	12	18	17	1
12	PAF 12070 P10	3	12	14	20	7	1
	PAF 12090 P10	3,7	12	14	20	9	1
	PAF 12120 P10	4,7	12	14	20	12	1
	PAF 12170 P10	6,1	12	14	20	17	1
14	PAF 14120 P10	5,4	14	16	22	12	1
	PAF 14170 P10	7,1	14	16	22	17	1
15	PAF 15090 P10	4,4	15	17	23	9	1
	PAF 15120 P10	5,7	15	17	23	12	1
	PAF 15170 P10	7,7	15	17	23	17	1
16	PAF 16120 P10	6	16	18	24	12	1
	PAF 16170 P10	8,3	16	18	24	17	1
18	PAF 18120 P10	6,7	18	20	26	12	1
	PAF 18170 P10	8,9	18	20	26	17	1
	PAF 18220 P10	11,1	18	20	26	22	1
20	PAF 20115 P10	11,6	20	23	30	11,5	1,5
	PAF 20165 P10	15,1	20	23	30	16,5	1,5
	PAF 20215 P10	19,1	20	23	30	21,5	1,5
25	PAF 25115 P10	14,2	25	28	35	11,5	1,5
	PAF 25165 P10	19	25	28	35	16,5	1,5
	PAF 25215 P10	23,9	25	28	35	21,5	1,5
30	PAF 30160 P10	30,9	30	34	42	16	2
	PAF 30260 P10	46,1	30	34	42	26	2
35	PAF 35160 P10	35,4	35	39	47	16	2
	PAF 35260 P10	52,7	35	39	47	26	2
40	PAF 40260 P10	60	40	44	53	26	2

Dimension table · Dimensions in mm							
Shaft diameter	Designation PAF..P11	Mass g	Dimensions				
			D _i	D _o	D _{FL} ±0,5	B ±0,25	S _{FL} -0,2
6	PAF 06080 P11	1,7	6	8	12	8	1
8	PAF 08055 P11	1,7	8	10	15	5,5	1
	PAF 08095 P11	2,5	8	10	15	9,5	1
10	PAF 10070 P11	2,5	10	12	18	7	1
	PAF 10120 P11	3,8	10	12	18	12	1
	PAF 10170 P11	5,4	10	12	18	17	1
12	PAF 12070 P11	3	12	14	20	7	1
	PAF 12090 P11	3,7	12	14	20	9	1
	PAF 12120 P11	4,7	12	14	20	12	1
15	PAF 15120 P11	5,7	15	17	23	12	1
	PAF 15170 P11	7,7	15	17	23	17	1
16	PAF 16120 P11	6	16	18	24	12	1
18	PAF 18100 P11	5	18	20	26	10	1
	PAF 18220 P11	11,1	18	20	26	22	1
20	PAF 20115 P11	11,6	20	23	30	11,5	1,5
	PAF 20165 P11	15,1	20	23	30	16,5	1,5
25	PAF 25215 P11	23,9	25	28	35	21,5	1,5
30	PAF 30160 P11	30,9	30	34	42	16	2
	PAF 30260 P11	46,1	30	34	42	26	2
35	PAF 35260 P11	52,7	35	39	47	26	2
40	PAF 40260 P11	59	40	44	53	26	2

Recommended mounting tolerance:

Shaft f7 Housing bore H7

Internal clearances, wall thicknesses and chamfer tolerances: see page 25ff.

Bushes in special sizes are available by agreement.

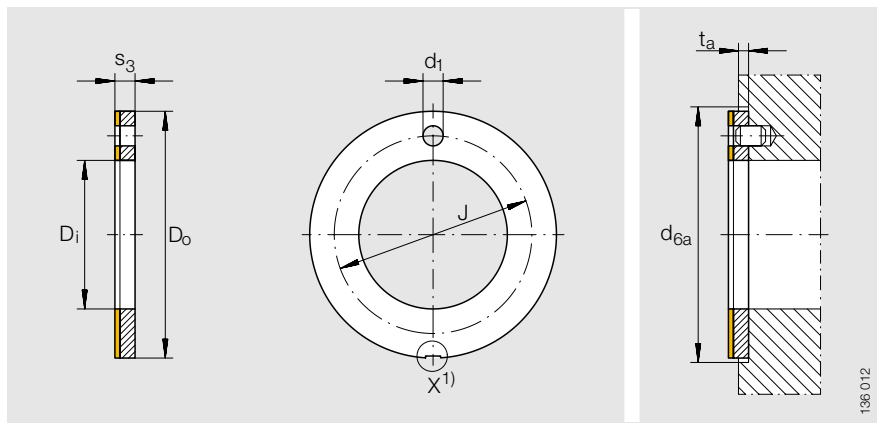
1) Inside diameter Radius
D_i ≤ 8: R1-0,5
D_i > 8: R1±0,5

Permaglide®

Thrust washers

Maintenance-free,
with steel backing

Series PAW..P10



PAW

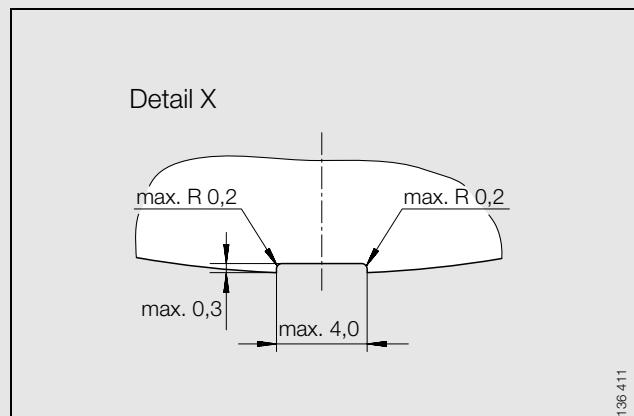
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Dimension table · Dimensions in mm

Designation	Mass g	Dimensions					Mounting dimensions		
		Di +0,25	Do -0,25	s3 -0,05	J ±0,12	d1 +0,4 +0,1	ta ±0,2	d6a +0,12	
PAW 10 P10	2,6	10	20	1,5	15	1,5	1	20	
PAW 12 P10	3,8	12	24	1,5	18	1,5	1	24	
PAW 14 P10	4,3	14	26	1,5	20	2	1	26	
PAW 16 P10	5,5	16	30	1,5	22	2	1	30	
PAW 18 P10	6	18	32	1,5	25	2	1	32	
PAW 20 P10	8	20	36	1,5	28	3	1	36	
PAW 22 P10	8,5	22	38	1,5	30	3	1	38	
PAW 26 P10	11,1	26	44	1,5	35	3	1	44	
PAW 28 P10	13,3	28	48	1,5	38	4	1	48	
PAW 32 P10	16,3	32	54	1,5	43	4	1	54	
PAW 38 P10	21	38	62	1,5	50	4	1	62	
PAW 42 P10	23,5	42	66	1,5	54	4	1	66	
PAW 48 P10	36,8	48	74	2	61	4	1,5	74	
PAW 52 P10	38,8	52	78	2	65	4	1,5	78	
PAW 62 P10	48,8	62	90	2	76	4	1,5	90	

Thrust washers in special sizes are available by agreement.
Thrust washers PAW..P11 are available by agreement.

1) A maximum of 4 cutouts are permissible on the outside diameter in any position.



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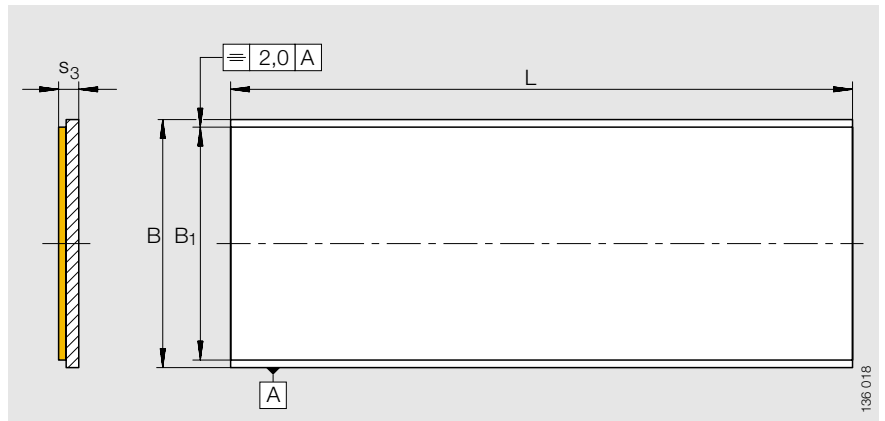


Permaglide®

Strips

Maintenance-free,
with steel backing
Series PAS..P10

Maintenance free,
with bronze backing
Series PAS..P11



PAS..P10, PAS..P11

Dimension table · Dimensions in mm					
Designation PAS..P10	Mass g	Dimensions			
		s ₃	B	B ₁	L
		-0,04	+1,5		+3
PAS 05180 P10	330	0,5	180	168	500
PAS 07180 P10¹⁾	506	0,75	180	168	500
PAS 07250 P10²⁾	703	0,75	250	238	500
PAS 10250 P10	939	1	250	238	500
PAS 15250 P10	1 421	1,5	250	238	500
PAS 20250 P10	1 914	2	250	238	500
PAS 25250 P10	2 392	2,5	250	238	500
PAS 30250 P10	2 940	3,06	250	238	500

Strips in special sizes are available by agreement.

B = Total width

B₁ = Usable width

¹⁾ Type to be discontinued; conversion to PAS 07250 P10 after existing supplies have been sold off.

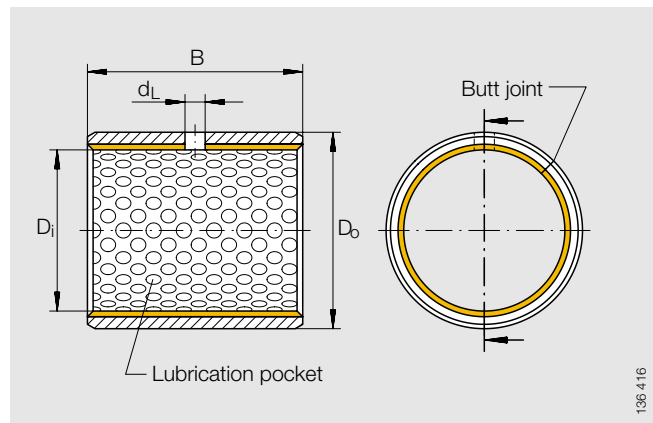
²⁾ Available only once stocks of PAS 07180 P10 have been sold off.

Dimension table · Dimensions in mm					
Designation PAS..P11	Mass g	Dimensions			
		s ₃	B	B ₁	L
		-0,04	+1,5		+3
PAS 10160 P11	663	1	160	148	500
PAS 15180 P11	1 131	1,5	180	168	500
PAS 20180 P11	1 523	2	180	168	500
PAS 25180 P11	1 915	2,5	180	168	500

Bushes

Low-maintenance

Series PAP..P20



PAP

Dimension table · Dimensions in mm						
Shaft diameter	Designation	Mass g	Dimensions			
			D _i	D _o	B ±0,25	d _L
8	PAP 0808 P20	1,6	8	10	8	- ¹⁾
	PAP 0810 P20	2	8	10	10	- ¹⁾
	PAP 0812 P20	2,4	8	10	12	- ¹⁾
10	PAP 1008 P20	1,8	10	12	8	- ¹⁾
	PAP 1010 P20	2,3	10	12	10	3
	PAP 1015 P20	3,5	10	12	15	3
12	PAP 1210 P20	3	12	14	10	3
	PAP 1212 P20	3,5	12	14	12	3
	PAP 1215 P20	4,4	12	14	15	3
	PAP 1220 P20	6	12	14	20	3
14	PAP 1420 P20	6,7	14	16	20	3
15	PAP 1510 P20	3,5	15	17	10	3
	PAP 1515 P20	5,3	15	17	15	3
	PAP 1525 P20	8,8	15	17	25	3
16	PAP 1612 P20	4,5	16	18	12	3
	PAP 1615 P20	5,6	16	18	15	3
	PAP 1620 P20	7,5	16	18	20	3
18	PAP 1815 P20	6,3	18	20	15	3
	PAP 1820 P20	8,5	18	20	20	3
20	PAP 2015 P20	10,5	20	23	15	3
	PAP 2020 P20	14	20	23	20	3
	PAP 2025 P20	17,5	20	23	25	3
	PAP 2030 P20	23	20	23	30	3
22	PAP 2220 P20	15,4	22	25	20	3
25	PAP 2515 P20	14,2	25	28	15	4
	PAP 2520 P20	19	25	28	20	4
	PAP 2525 P20	22	25	28	25	4
	PAP 2530 P20	26,4	25	28	30	4
28	PAP 2830 P20	40	28	32	30	4
30	PAP 3020 P20	28,6	30	34	20	4
	PAP 3025 P20	35,8	30	34	25	4
	PAP 3030 P20	42,9	30	34	30	4
	PAP 3040 P20	57	30	34	40	4
32	PAP 3230 P20	48	32	36	30	4

Dimension table (continued) · Dimensions in mm						
Shaft diameter	Designation	Mass g	Dimensions			
			D _i	D _o	B ±0,25	d _L
35	PAP 3520 P20	33	35	39	20	4
	PAP 3530 P20	49	35	39	30	4
	PAP 3550 P20	88	35	39	50	4
40	PAP 4020 P20	37	40	44	20	4
	PAP 4030 P20	56	40	44	30	4
	PAP 4040 P20	75	40	44	40	4
	PAP 4050 P20	93	40	44	50	4
45	PAP 4540 P20	113	45	50	40	5
	PAP 4550 P20	133	45	50	50	5
50	PAP 5025 P20	78	50	55	25	5
	PAP 5040 P20	118	50	55	40	5
	PAP 5060 P20	176	50	55	60	5
55	PAP 5540 P20	137	55	60	40	5
60	PAP 6030 P20	104	60	65	30	6
	PAP 6040 P20	139	60	65	40	6
	PAP 6060 P20	209	60	65	60	6
70	PAP 7040 P20	173	70	75	40	6
	PAP 7050 P20	202	70	75	50	6
	PAP 7070 P20	303	70	75	70	6
75	PAP 7540 P20	184	75	80	40	6
	PAP 7580 P20	370	75	80	80	6
80	PAP 8040 P20	184	80	85	40	6
	PAP 8055 P20	271	80	85	55	6
	PAP 8060 P20	276	80	85	60	6
90	PAP 8080 P20	394	80	85	80	6
	PAP 9060 P20	310	90	95	60	6
100	PAP 10050 P20	285	100	105	50	8
	PAP 10060 P20	343	100	105	60	8

Recommended mounting tolerance:
Shaft h8 Housing bore H7

Internal clearances, wall thicknesses and chamfer tolerances:
see page 25ff.
Deformation of lubrication hole due to round bending is permissible.
Bushes in special sizes are available by agreement.

¹⁾ No lubrication hole.

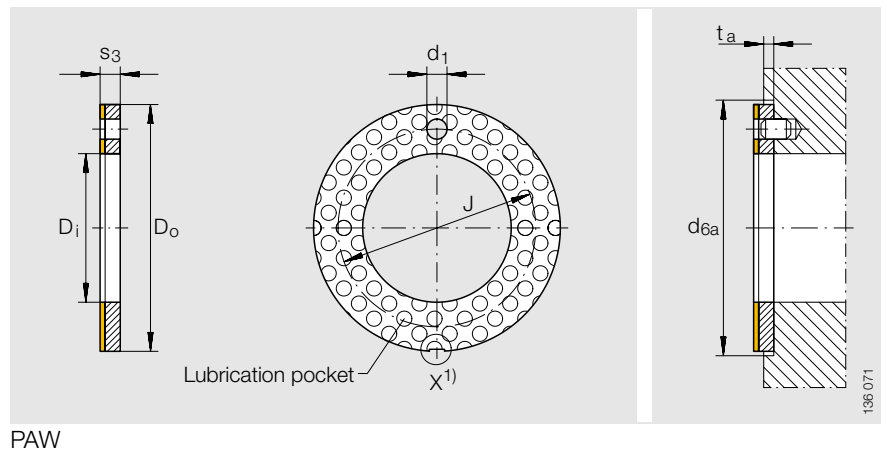


Permaglide®

Thrust washers

Low-maintenance

Series PAW..P20



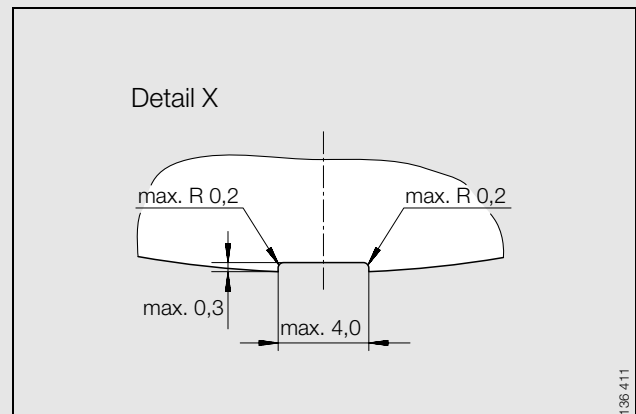
PAW

Dimension table · Dimensions in mm

Designation	Mass g	Dimensions					Mounting dimensions	
		Di +0,25	Do -0,25	s3 -0,05	J ±0,12	d1 +0,4 +0,1	ta ±0,2	d6a +0,12
PAW 12 P20	3,5	12	24	1,5	18	1,5	1	24
PAW 14 P20	3,7	14	26	1,5	20	2	1	26
PAW 18 P20	5,6	18	32	1,5	25	2	1	32
PAW 20 P20	7,4	20	36	1,5	28	3	1	36
PAW 22 P20	7,9	22	38	1,5	30	3	1	38
PAW 26 P20	10,3	26	44	1,5	35	3	1	44
PAW 28 P20	12,4	28	48	1,5	38	4	1	48
PAW 32 P20	15,2	32	54	1,5	43	4	1	54
PAW 38 P20	19,5	38	62	1,5	50	4	1	62
PAW 42 P20	20,8	42	66	1,5	54	4	1	66
PAW 48 P20	34,9	48	74	2	61	4	1,5	74
PAW 52 P20	36,1	52	78	2	65	4	1,5	78

Thrust washers in special sizes are available by agreement.

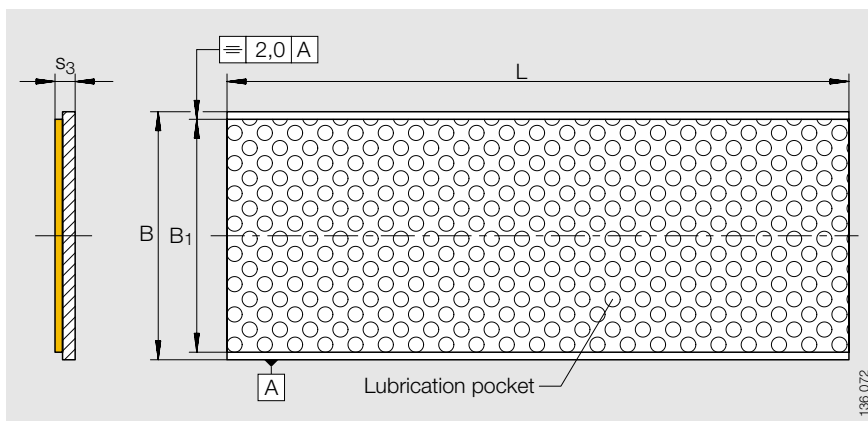
1) A maximum of 4 cut-outs are permissible on the outside diameter in any position.



Strips

Low-maintenance

Séries PAS..P20
PAS..P21
PAS..P22



PAS..P20, with lubrication pockets

PAS..P21, with machining allowance and lubrication pockets

Dimension table · Dimensions in mm					
Designation PAS..P20	Mass g	Dimensions			
		s_3	B	B_1	L
		-0,04	+1,5		+3
PAS 10180 P20	640	0,99	180	168	500
PAS 15180 P20	951	1,48	180	168	500
PAS 20180 P20	1 281	1,97	180	168	500
PAS 25180 P20	1 602	2,46	180	168	500

B = Total width

B_1 = Usable width

Strips in special sizes are available by agreement.

Dimension table · Dimensions in mm					
Designation PAS..P21	Mass g	Dimensions			
		$s_3^{1)}$	B	B_1	L
		-0,04	+1,5		+3
PAS 10180 P21	711	1,11	180	168	500
PAS 15180 P21	990	1,61	180	168	500
PAS 20180 P21	1 320	2,11	180	168	500
PAS 25180 P21	1 641	2,63	180	168	500

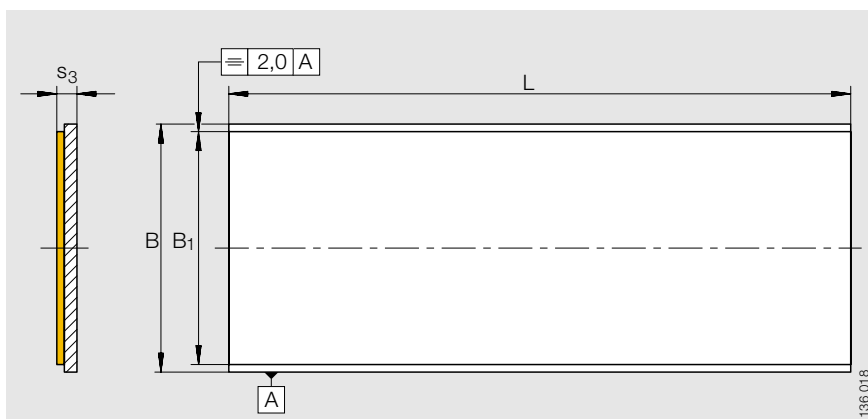
Available by agreement.

1) Machining allowance: 0,15 mm.

Dimension table · Dimensions in mm					
Designation PAS..P22	Mass g	Dimensions			
		$s_3^{1)}$	B	B_1	L
		-0,04	+1,5		+3
PAS 10180 P22	711	1,11	180	168	500
PAS 15180 P22	990	1,61	180	168	500
PAS 20180 P22	1 320	2,11	180	168	500
PAS 25180 P22	1 641	2,63	180	168	500

Available by agreement.

1) Machining allowance: 0,15 mm.



PAS..P22, with machining allowance, without lubrication pockets



Special designs

Linear plain bearings

In addition to the products presented in this catalogue, numerous special designs are available:

- made from any Permaglide® plain bearing material
- in non-standard dimensions
- as combined components ①, ②
 - pressed into rings
 - with plastic outer tyres
- in different shapes ⑦, ⑭
 - bushes with windows and holes ③, ⑤
 - bushes with stamped oil grooves ④, ⑥
 - blanked parts ⑬, ⑮, ⑯
 - ball cups ⑩, ⑪, ⑫
 - bearing shells ⑰, ⑱
- with an external sliding layer ⑧, ⑨
- with different butt joint forms ⑨.

A small selection of special components already produced is shown here.

Special designs can be made in the following sizes:

- with bush outside diameters between 3 mm and 305 mm (in special cases up to 800 mm)
- with a strip width up to 250 mm
- with wall thicknesses from 0,5 to 3,06 mm.

⚠ The feasibility of special designs should be confirmed as early as possible. This applies not only to the geometry but also to costs. Please contact INA for further information.

Permaglide® linear plain bearings

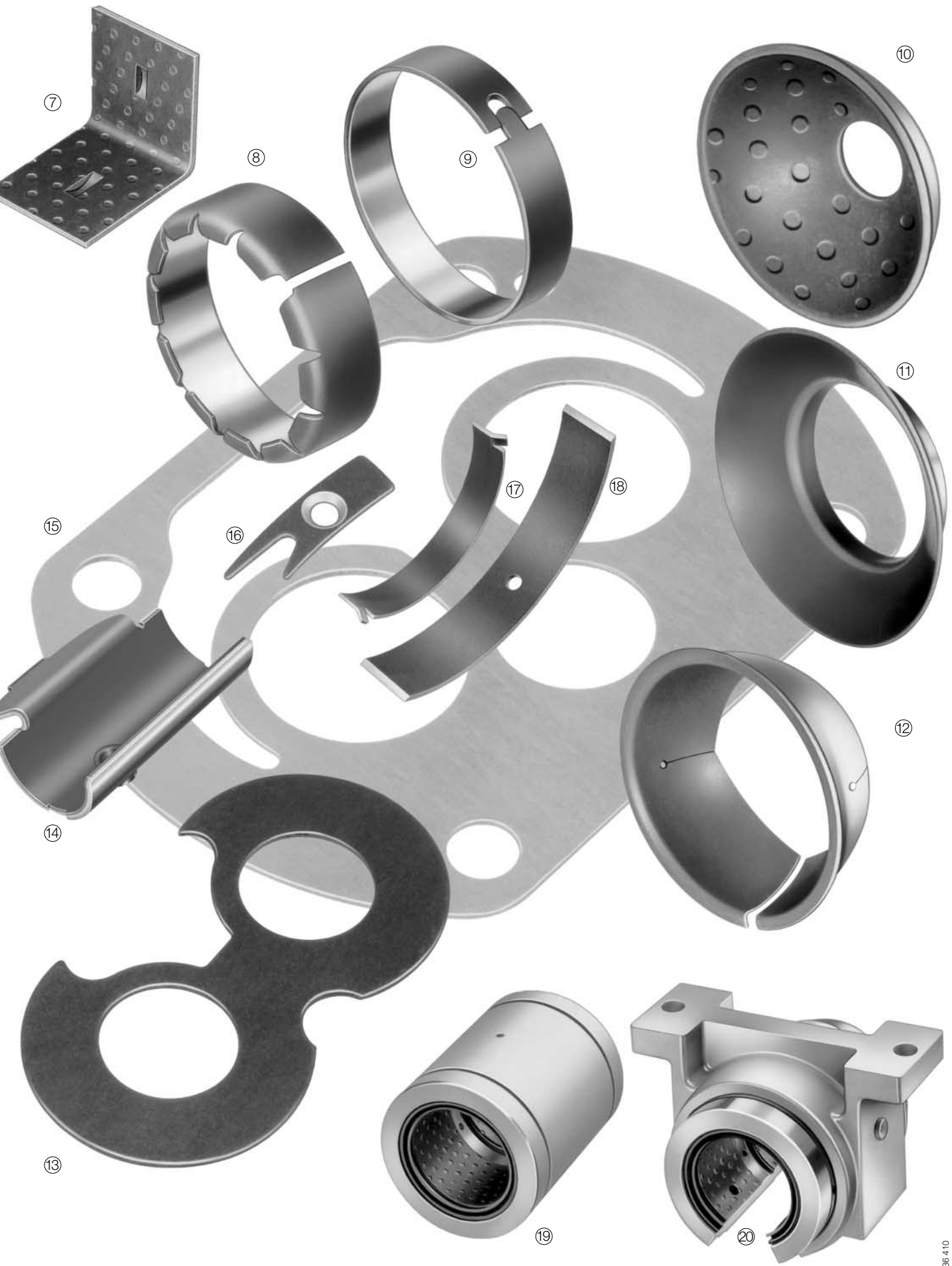
Permaglide® linear plain bearings PAB consist of Permaglide® bushes PAP..P20 ⑰ pressed into an outer ring. The PABO design has a segment cut out of the bearing for use with supported shafts.

Permaglide® linear plain bearing units PAGH and PAGBA consist of a Permaglide® linear plain bearing PAB or PABO ⑰ pressed into a housing.



 Further information on Permaglide® linear plain bearings: INA publication LIF "Linear Guidance Systems", CD ROM "medias® professional"

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Each section of Catalogue 706 starts with a description of the product series and product features.

The significant characteristics of the products are indicated by means of pictograms.

The advantages of this type of presentation include:

- less reading is required
- the required information is accessed more quickly
- a direct comparison between alternative products is possible.

Definition of pictograms

Pictogram	Description
	The products can support radial loads
	The products can support axial loads
	If this information is not adhered to, there is a direct or indirect risk to the product and/or the adjacent construction.